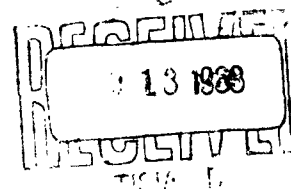


NAVY DEPARTMENT  
THE DAVID W. TAYLOR MODEL BASIN  
AERODYNAMICS LABORATORY  
WASHINGTON 7, D.C.

TEST RESULTS OF A TWO-DIMENSIONAL PLENUM CHAMBER GROUND  
EFFECT MACHINE OVER A LIQUID SURFACE

by

Anibal Alfredo Tinajero



Qualified requesters may obtain copies of this report  
direct from ASTIA

410027

June 1963

CATALOGED BY DDC 410027

AS AD No. \_\_\_\_\_

**FOR ERRATA**

**AD \_\_\_\_\_**  
**410027**

**THE FOLLOWING PAGES ARE CHANGES**

**TO BASIC DOCUMENT.**

AD 410 027

410027

# ERRATA

DAVID TAYLOR MODEL BASIN AERO REPORT 1052

By Anibal Alfredo Tinajero

June 1963

The following changes should be made to the report:

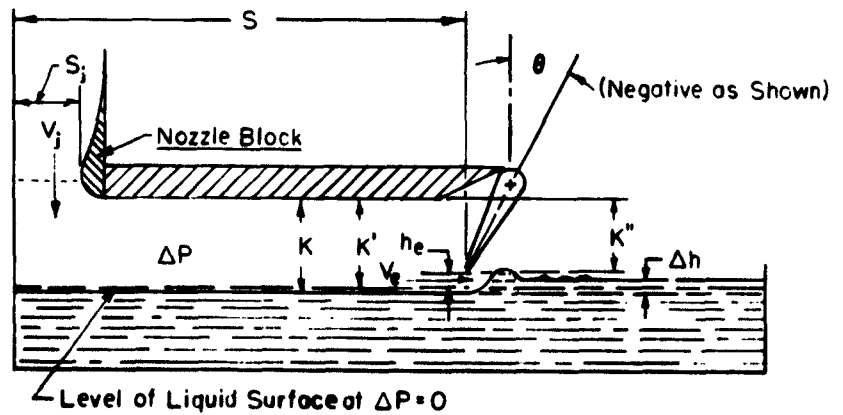
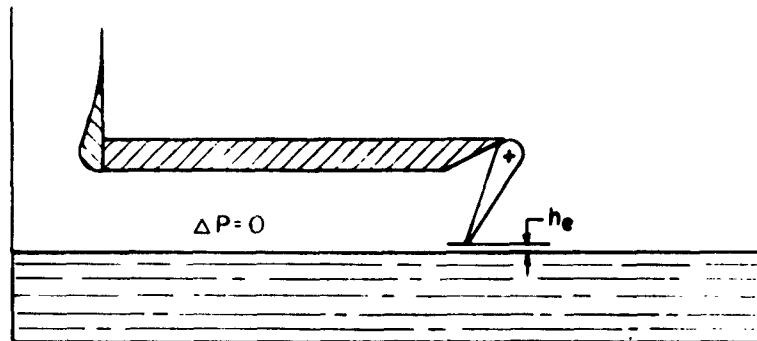
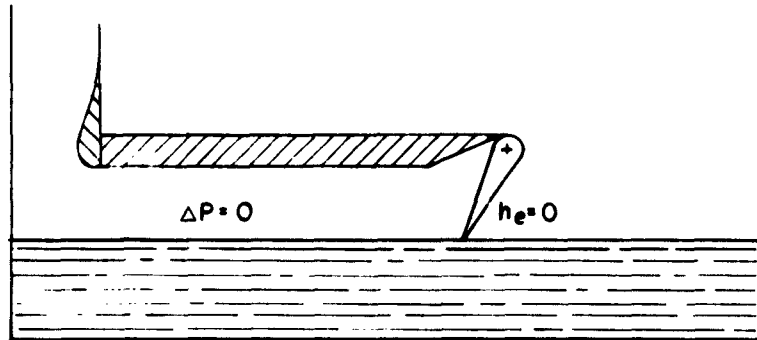
Under "Symbols" delete the last definition.

Page 3: The symbol  $\rho$  instead of  $\rho'$  should appear in equation [2] and in the line following the equation.

Page 3, last line: The following should be deleted: "(water, in this case)."

Issued 18 July 1963  
DTMB - Washington 7, D. C.

# NOTATION



# SYMBOLS

$a_t$	speed of sound in air ( $a_t = 1117$ feet per second)
$C$	perimeter of flap or skirt (taken at lowest point) in feet
$C_p$	specific heat at constant pressure in Btu/pound-degree Fahrenheit
$C_v$	specific heat at constant volume in Btu/pound-degree Fahrenheit
$D_c$	discharge coefficient
$h$	height of lowest point of flap or skirt to depressed liquid surface in feet ( $h = k - k''$ ) = $h_e + \frac{\Delta h}{2}$ for report data only
$\Delta h$	amount of liquid surface depression in feet
$h_e$	height of lowest point of flap or skirt from free liquid surface (when $\Delta h = 0$ ) in feet
$J$	jet momentum force in pounds
$k$	height of model base from depressed surface of liquid in feet ( $k = k' + \frac{\Delta h}{2}$ for test)
$k'$	vertical distance of model base to free surface of liquid when $\Delta h = 0$ in feet
$k''$	vertical distance of model base to lowest point of flap or skirt in feet ( $k'' = k' - h_e$ )
$P$	atmospheric pressure ( $P = 14.7$ pounds per square inch)
$\Delta P$	base pressure in pounds per square foot
$P_u$	air supply reference pressure in pounds per square inch
$Q$	rate of flow in cubic feet per second
$r$	distance from edge of flap to water surface at minimum escape area (see Figure 15)
$S$	total planform area in square feet
$S_i$	intake area in square feet
$S_e$	escape area in square feet ( $S_e = Ch$ )
$V_e$	average escape velocity of air at point where $S_e$ is measured, in feet per second
$V_i$	average airspeed at intake of model in feet per second
$V_j$	average airspeed at flowmeter throat in feet per second
$\theta$	angle of inclination of flap or skirt in degrees (see notation)
$\rho$	density of air ( $\rho = 0.002378$ slug per cubic foot)
$\rho'$	density of water

AERODYNAMICS LABORATORY  
DAVID TAYLOR MODEL BASIN  
UNITED STATES NAVY  
WASHINGTON, D. C.

TEST RESULTS OF A TWO-DIMENSIONAL PLENUM-CHAMBER GROUND  
EFFECT MACHINE OVER A LIQUID SURFACE

by

Anibal Alfredo Tinajero

SUMMARY

A two-dimensional plenum-chamber GEM with a trailing-edge flap was tested over a liquid surface. The discharge coefficient was determined for the following parameters: ratio of intake to escape area, ratio of planform area to intake area, hover height, planform loading, flap angle, and distance from the trailing edge of the flap to the liquid.

INTRODUCTION

The present investigation was performed to determine the effects of intake area, planform loading, hover height, and flap (or skirt) angle upon the discharge coefficient of a two-dimensional plenum-chamber GEM.

The tests were conducted intermittently in the Aerodynamics Laboratory at the Taylor Model Basin during September and October, 1960. This investigation was carried out in accordance with Reference 1.

MODEL AND TEST EQUIPMENT

The test equipment consisted of a narrow rectangular tank with transparent sides. One end was enclosed with wood; the other was open to allow air to flow freely through a vertical nozzle. For simulation of flight

over a liquid surface, the open end required a barrier to retain the water. A photograph of the test setup is shown in Figure 1.

The model used for representing the GEM consisted of a board with a flap at one end and an inlet diffuser nozzle at the other end. The flap setting could be varied over a wide range of angles, and the size of the inlet was controlled by inserting different nozzle blocks. A photograph of a typical nozzle is shown in Figure 2. Static pressure orifices for determining pressures at the base of the model were located along the length of the board.

#### CALIBRATION OF FLOWMETER

The calibration of the flowmeter was carried out by measuring the jet force acting upon a modified chemical-type balance, for various values of air supply pressure  $P_u$ . A photograph of the flowmeter is shown in Figure 3. The procedure for calibration was to discharge the air through the flowmeter perpendicular to the balance plate. For values of  $P_u$  from 10 to 90 psi, the force  $J$  on the balance was recorded. The height of the nozzle was that for which the balance would give the highest reading.

The air volume rate of flow,  $Q$ , was calculated from the following equation:

$$Q = J/\rho V_j \quad [1]$$

For several ratios of  $P/P_u$  (assuming  $P = 14.7$  psi and the speed of sound  $a_t = 1117$  feet per second), corresponding ratios of  $V_j/a_t$  were found in the tables presented in Reference 2 (for  $C_p/C_v = 1.4$ ). For practical purposes, the value of  $\rho$  in Equation [1] was assumed constant at 0.002378 slug per cubic foot.

The calibration of the flowmeter, therefore, was in terms of rate of flow for air-supply pressure.

#### TEST PROCEDURE

Data were obtained at selected ratios of inlet area to model base area, various flap angle settings, ratios of  $h_e/k'$ , and pressure. The tests were conducted in two phases.

In the first phase (with no air flowing through the flowmeter), the free water surface was barely in contact with the trailing edge of the flap; i.e.,  $h_e$  was equal to 0. The values for the flap angle  $\theta$  were varied from  $-75^\circ$  to  $75^\circ$  in  $15^\circ$  increments. Then values of  $\Delta h$  were recorded for corresponding values of  $P_u$ . The static pressure orifices were utilized to determine whether there was a static pressure gradient along the length of the model and to check the accuracy in the measurement of depth of depression of the water. In a sense, one might look at the model and flow stand as a giant U-tube manometer. Because of the geometry of the model and by definition of  $h_e$ , it was necessary to vary the amount of water in the tank in order to maintain the height  $h_e$  constant at 0. The same procedure was repeated for each of the inlet nozzle blocks.

For the second phase,  $h_e$  was set at some value other than zero. This was done either by adding or evacuating water from the test stand. The values of  $k'$  changed automatically with a change in  $\theta$ , as shown in the notation. The same procedure was repeated for each inlet condition.

### RESULTS

The effect of the ratios  $S_i/S_e$ ,  $h/k'$ ,  $\Delta h/h$ ,  $h_e/k$ , and  $\theta$  upon the discharge coefficient  $D_c$  is presented in Figures 4 through 14. The physical meaning of these quantities can be better understood after a careful study of the test setup and the sketch in the notation.

The discharge coefficient  $D_c$  was computed from

$$D_c = \frac{Q}{S_e \sqrt{\frac{2\Delta P}{\rho}}} \quad [2]$$

where  $\Delta P$  was obtained from the pressure head,  $\Delta h$ , and  $\rho'$  is the density of the fluid (water, in this case).



## DISCUSSION

Because of the small size of the test stand and the crude nature of the equipment which was used to obtain the results, the data should be used for preliminary evaluation only. Further testing should be done on a larger scale model, in order to substantiate the results presented herein. For some conditions during the tests it was impossible to obtain data points at the higher values of mass flow, because of an undamped wave-producing oscillation. In some instances (high  $\Delta h$ ), the re-enforcement of the oscillation was sufficient to practically evacuate the water from the tank.

A careful study of the data will show that for constant values of  $k'$  (with  $\theta$  constant), if  $h$  increases (because of an increase in  $\Delta h$ ), the discharge coefficient is decreased. That is, as the planform loading increases, the discharge coefficient decreases. This effect depends on the nature of the flow at the flap trailing edge. To obtain true values of the discharge coefficients for plenum GEM with a flap or skirt, the exhaust area should be computed from

$$S_e = Cr \quad [3]$$

as shown in Figure 15.

For the test results presented in this report,  $h$  was used instead of  $r$ , since it was impossible to determine the value of  $r$  with sufficient accuracy on a model of such small scale. The value of  $h$  was computed from  $h = h_e + \frac{\Delta h}{2}$ . This was done to simplify testing, since the average position of the flap trailing edge was approximately at the center of the test stand. A slight error due to a small variation in total planform area  $S$  occurred for high negative or positive values of  $\theta$ . Also, blocking of the escape area occurs at high values of planform loading (high value of  $\Delta h$ ), because of spillage from the wave generated by the air as it leaves the trailing edge of the flap. For this same configuration, spray is generated by the escaping air travelling in a path tangential to the liquid surface. The amount of spray decreases directly with decrease in  $\Delta h$ , and the nature of the spray was found to vary with  $\theta$ . This can be readily understood from the sketches in Figure 15. The escaping air

at the trailing edge of the flap is forced to travel through a curved path. The dynamic pressure of the escaping air creates a drop in static pressure (below atmospheric) on the flap surface. Equilibrium is maintained by the wave rise spilling slightly over and toward the flap. As the water spills over, the escaping air pushes against it, generating spray. This situation will be encountered in any GEM when hovering over a liquid surface. For a GEM cruising above "hump speed," the spray will not be generated as in hovering, since  $\Delta h = 0$ . The amount of lift created in hovering will be equal to the weight of the volume of liquid displaced.

#### CONCLUSIONS

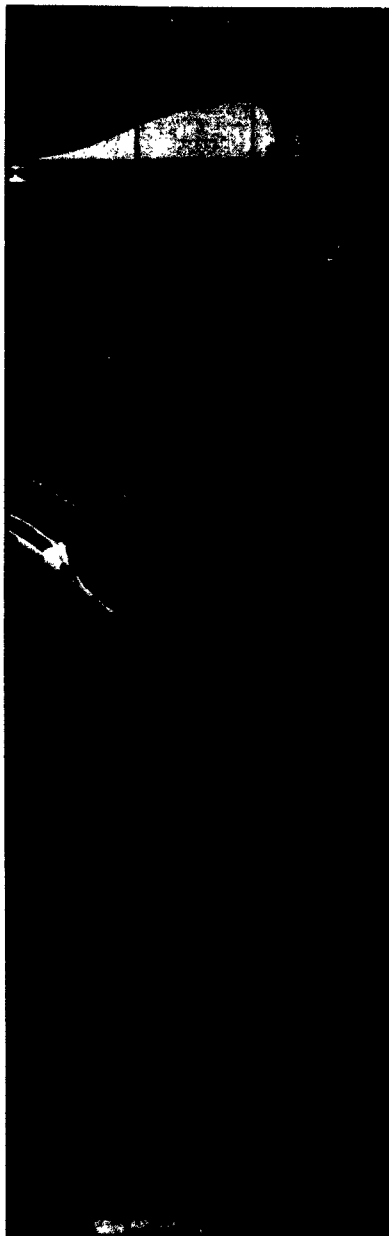
The results indicate that the discharge coefficient is a function of intake area, flap angle, and planform loading.

The values for discharge coefficients were found to vary considerably with planform loading. Discharge coefficient values should be considered only approximate, because of model size. These results do show a qualitative behavior of a plenum chamber GEM in the over-liquid hovering condition. Further tests, with a larger model, should be carried out to determine the exact contributions of each of the above-mentioned parameters to the discharge coefficients.

Aerodynamics Laboratory  
David Taylor Model Basin  
Washington, D. C.  
May 1963

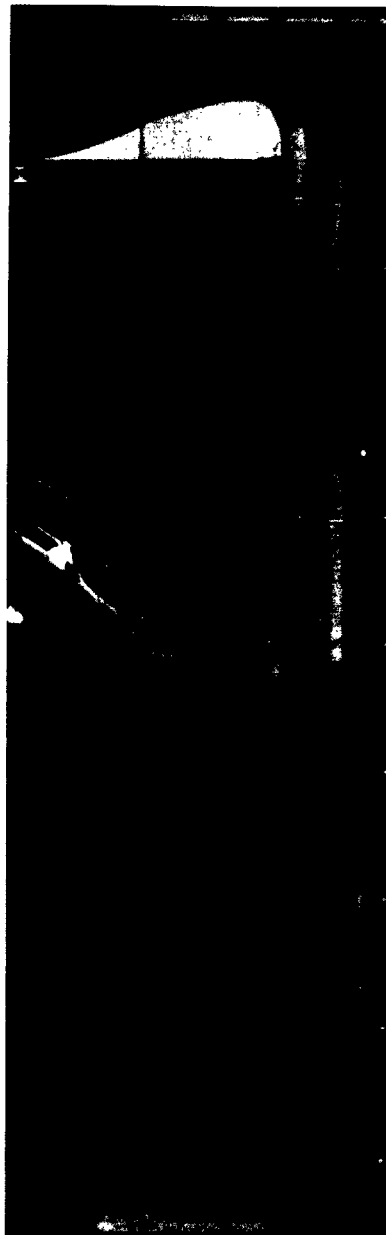
#### REFERENCES

1. BUWEPSINST 3920.1 R-12 of 2 Sep 1960
2. Bureau of Naval Weapons. Handbook of Supersonic Aerodynamics.  
Vol. 2. Wash., Oct 1950. 1 v. (loose-leaf) (NAWORD Rpt. 1488,  
Vol. 2)



PSD-304, 974

(a) Low Planform Loading



PSD-304, 975

(b) High Planform Loading

Figure 1 - Photographs of the Flow Stand

September 11, 1961



Figure 2 - Photograph of a Typical Intake Nozzle Block

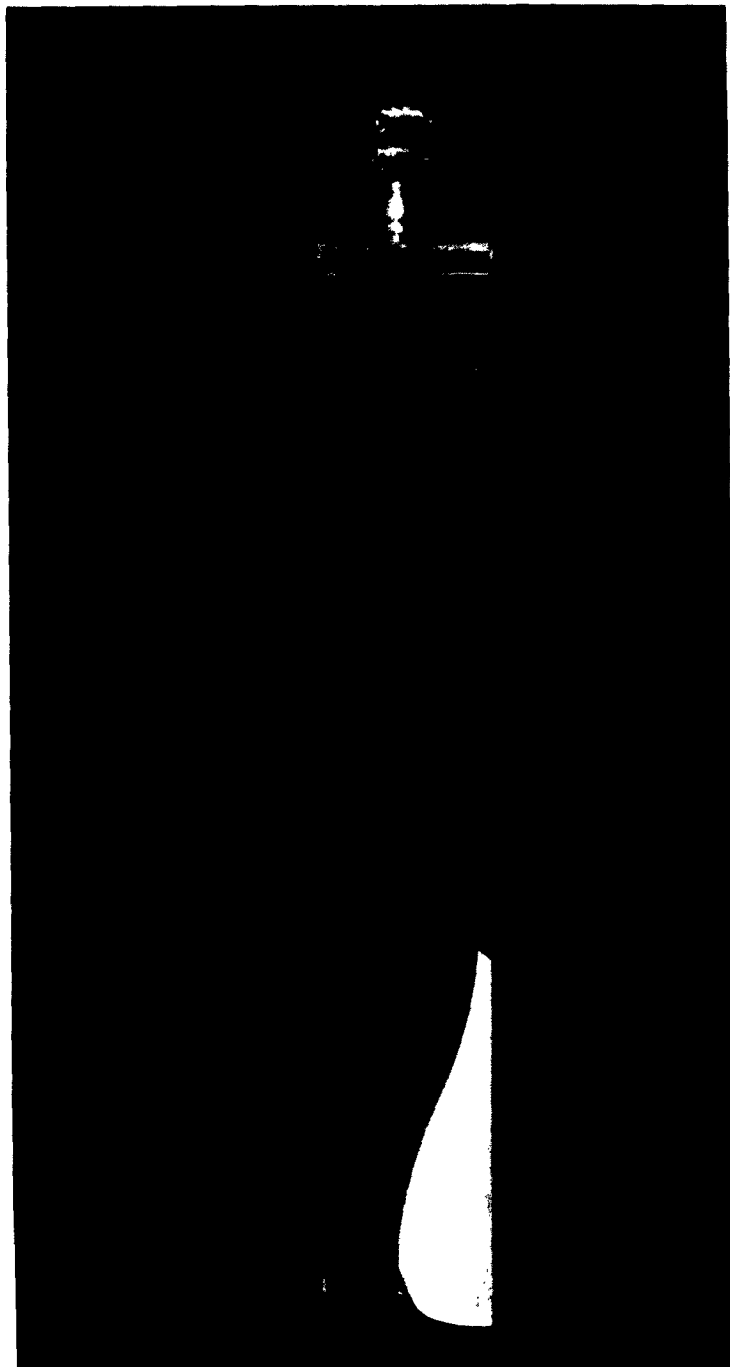


Figure 3 - Photograph of the Flowmeter

September 11, 1961

PSD-304,976

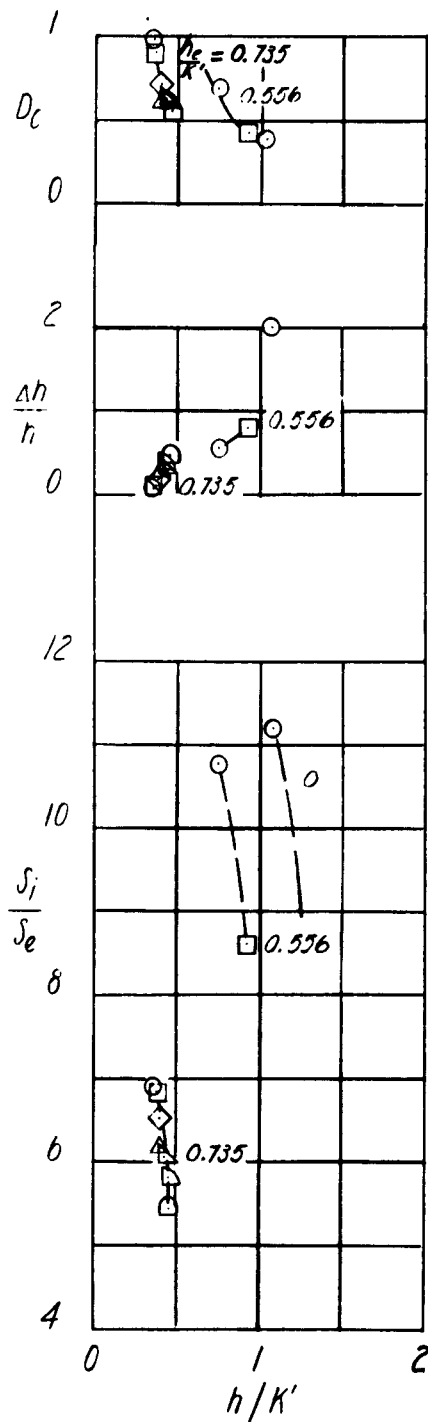


Figure 4 - Variation of the Parameters  $D_c$ ,  $\Delta h/h$ , and  $S_i/S_e$  With  $h/k'$  for Several Values of  $h_e/k'$ .  $\theta = 75^\circ$   
 (a)  $\frac{S}{S_1} = 6.078$

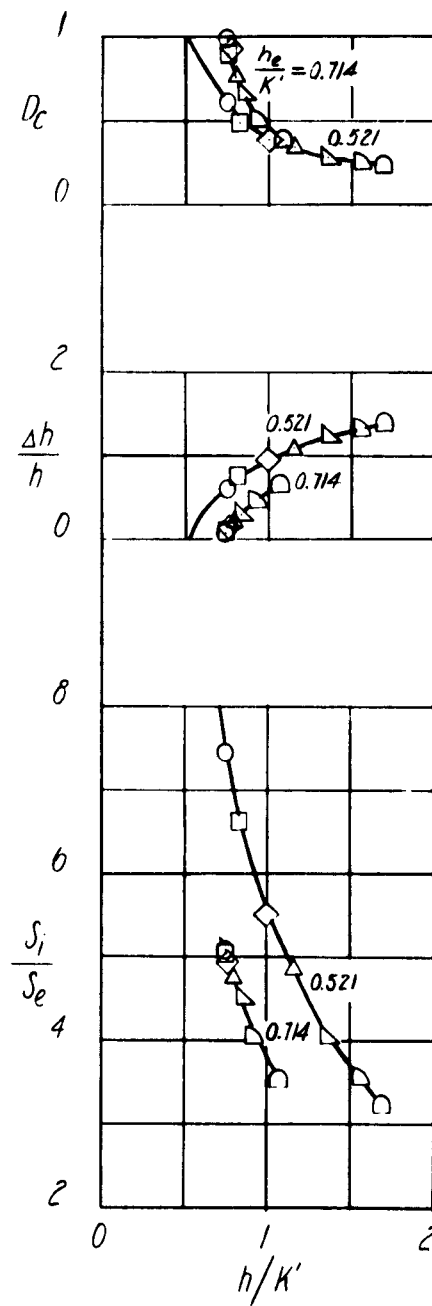


Figure 4 (Continued)

(b)  $\frac{S}{S_1} = 8.270$ 

FIGURE 4b

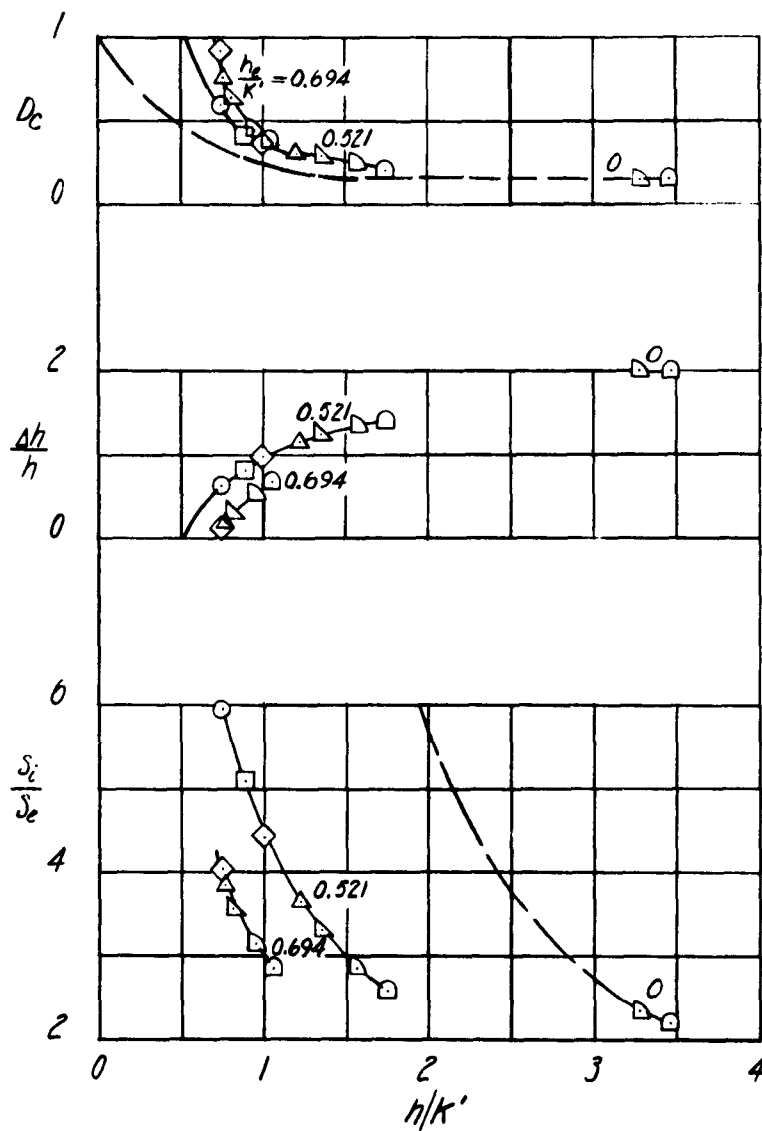


Figure 4 (Continued)

$$(c) \frac{s}{s_1} = 10.161$$

FIGURE 4c



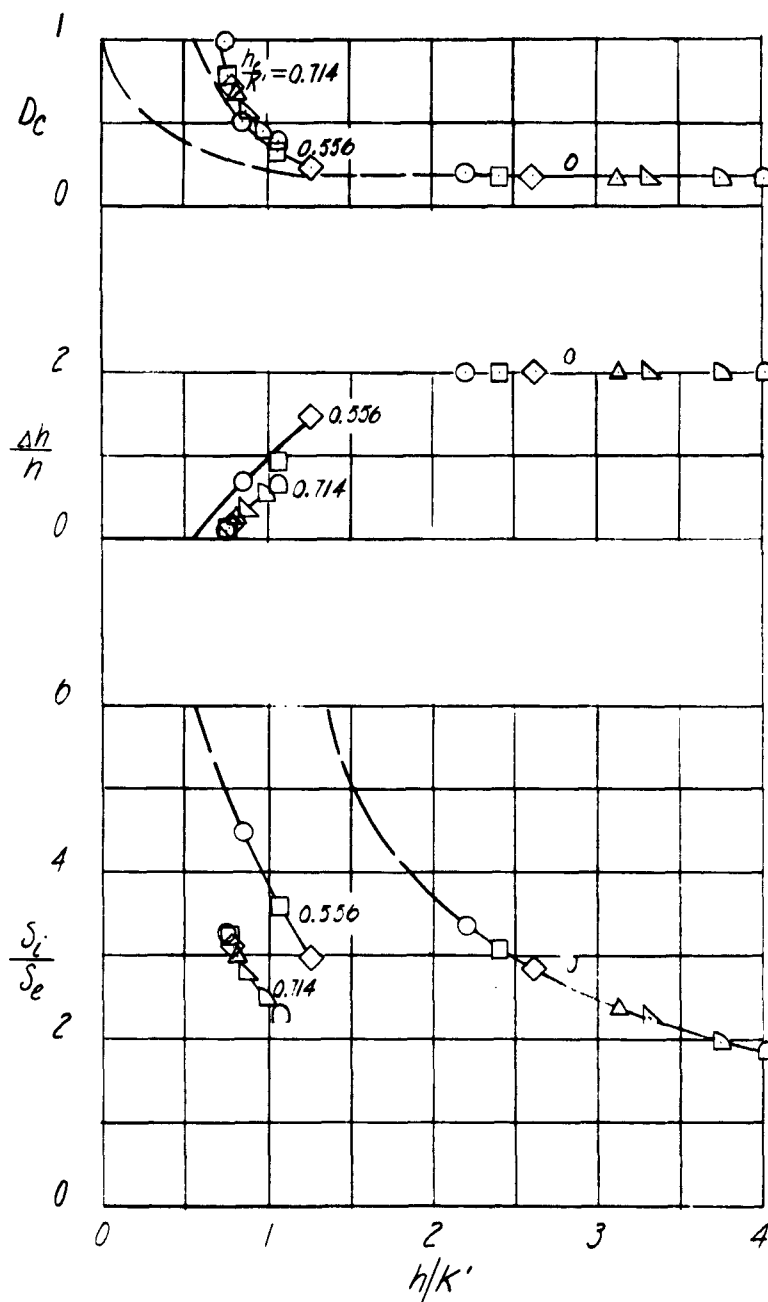


Figure 4 (Continued)

(d)  $\frac{S}{S_i} = 12.811$ 

FIGURE 4d

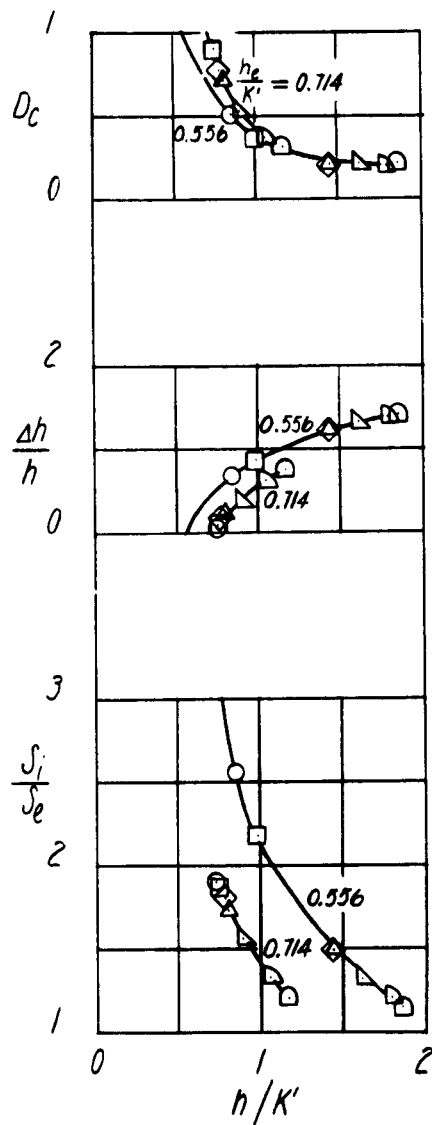


Figure 4 (Continued)

$$(*) \frac{S}{S_i} = 17.470$$

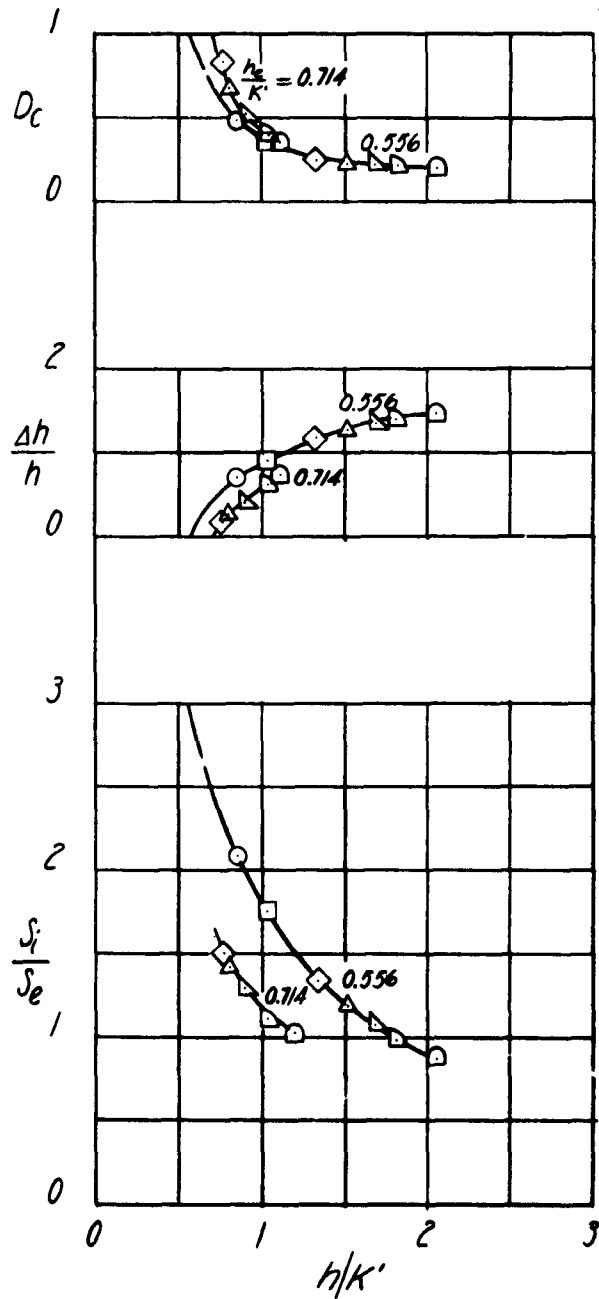


Figure 4 (Concluded)

$$(f) \frac{s_2}{s_1} = 27.111$$

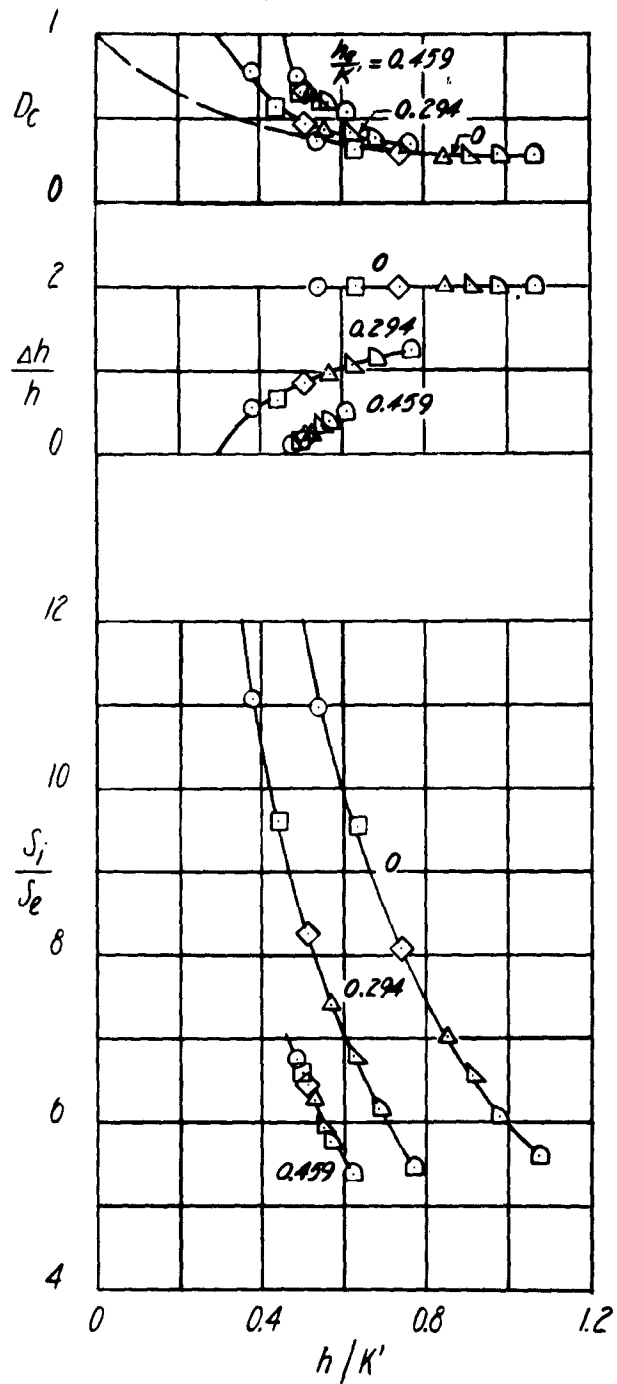


Figure 5 - Variation of the Parameters  $D_c$ ,  $\Delta h/h$ , and  $S_i/S_e$  With  $h/k'$  for Several Values of  $h_e/k'$ .  $\theta = 60^\circ$   
 (a)  $\frac{S}{S_1} = 6.036$

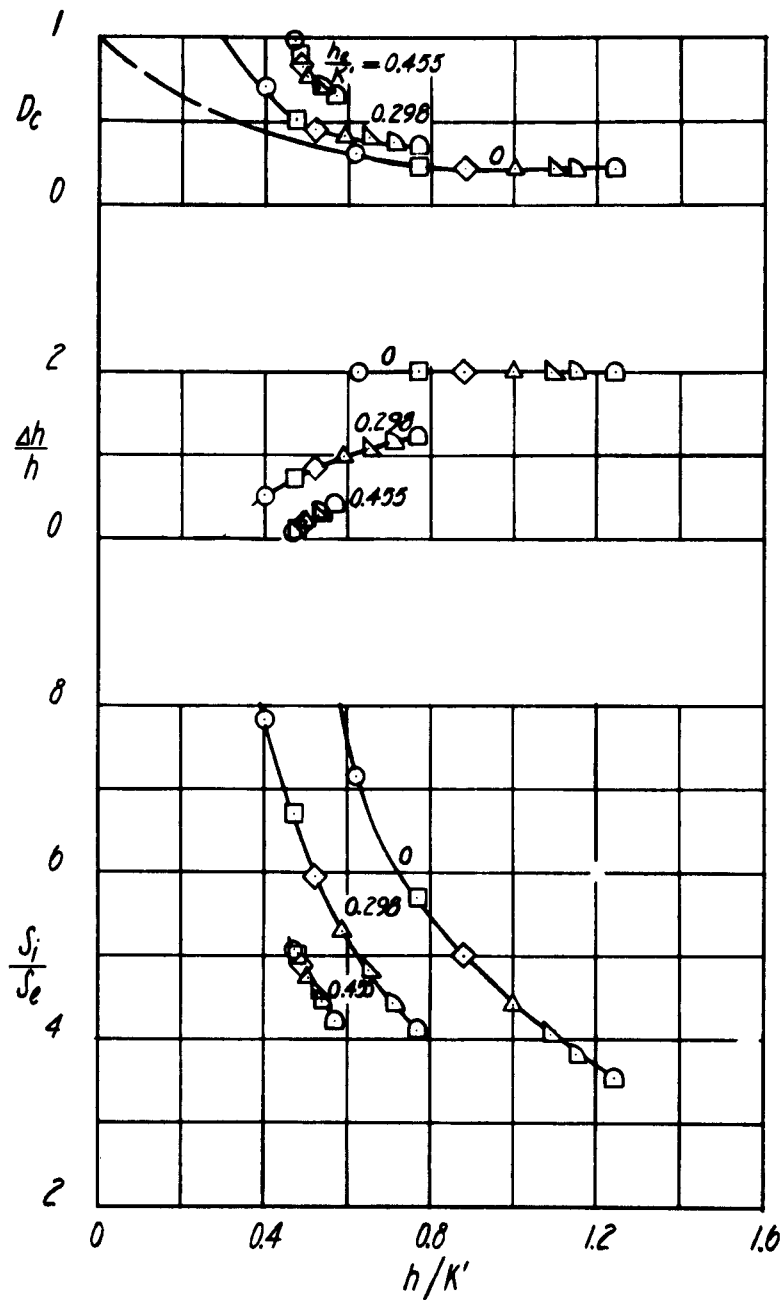


Figure 5 (Continued)

$$(b) \frac{S}{S_1} = 8.214$$

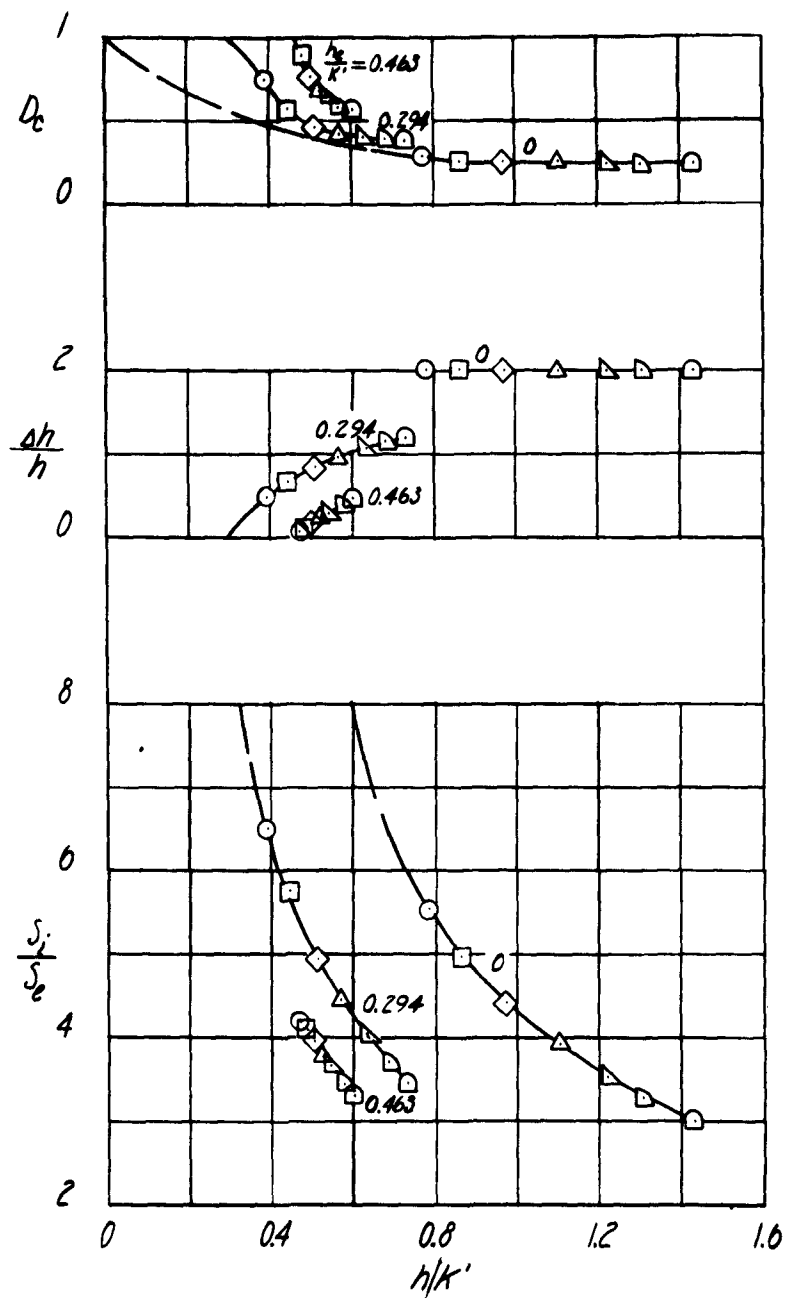


Figure 5 (Continued)

(c)  $\frac{s_2}{s_1} = 10.091$ 

FIGURE 5c

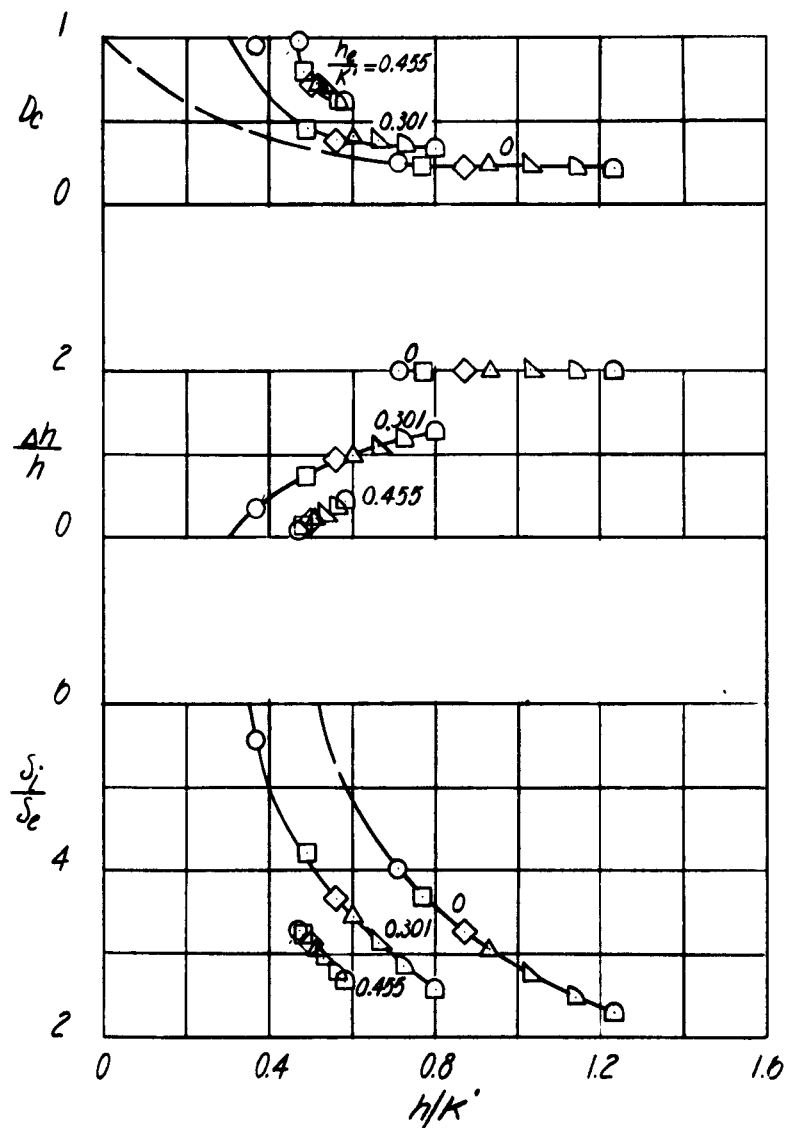


Figure 5 (Continued)  
 (d)  $\frac{s}{s_1} = 12.723$

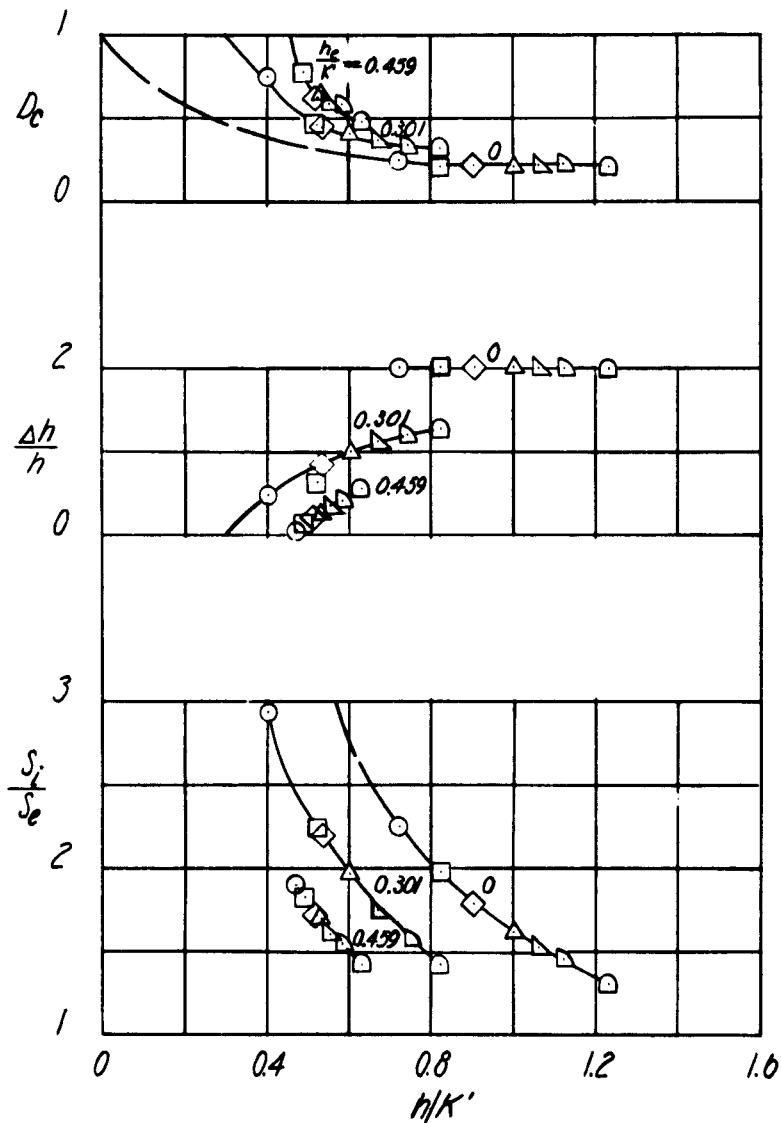


Figure 5 (Continued)

$$(e) \frac{S}{S_1} = 17.349$$



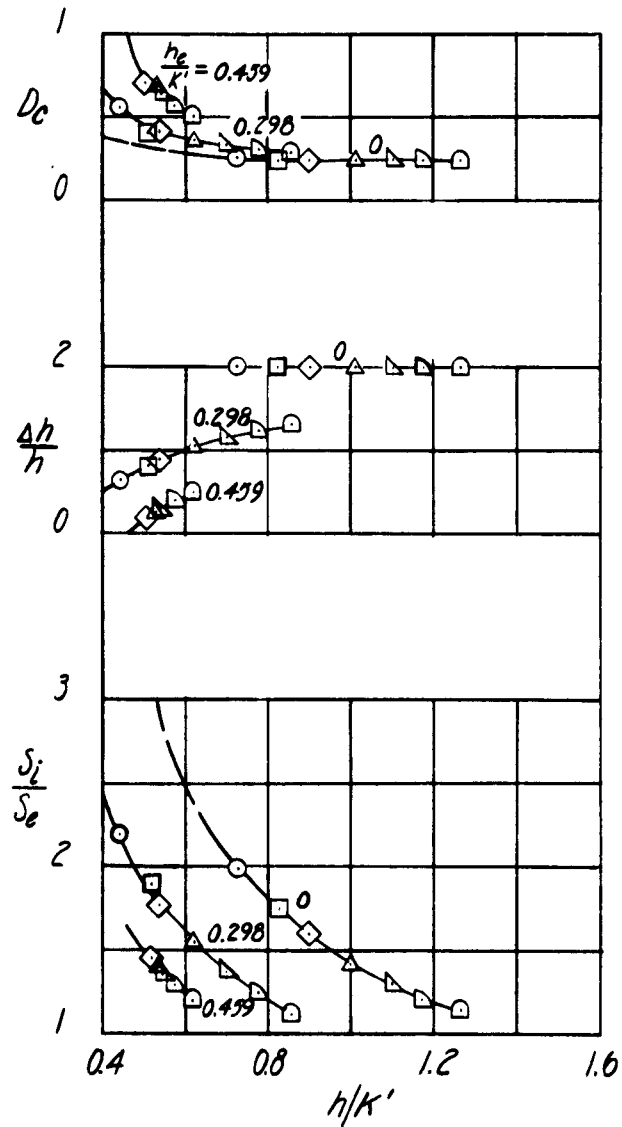


Figure 5 (Concluded)

$$(f) \frac{S}{S_1} = 26.924$$

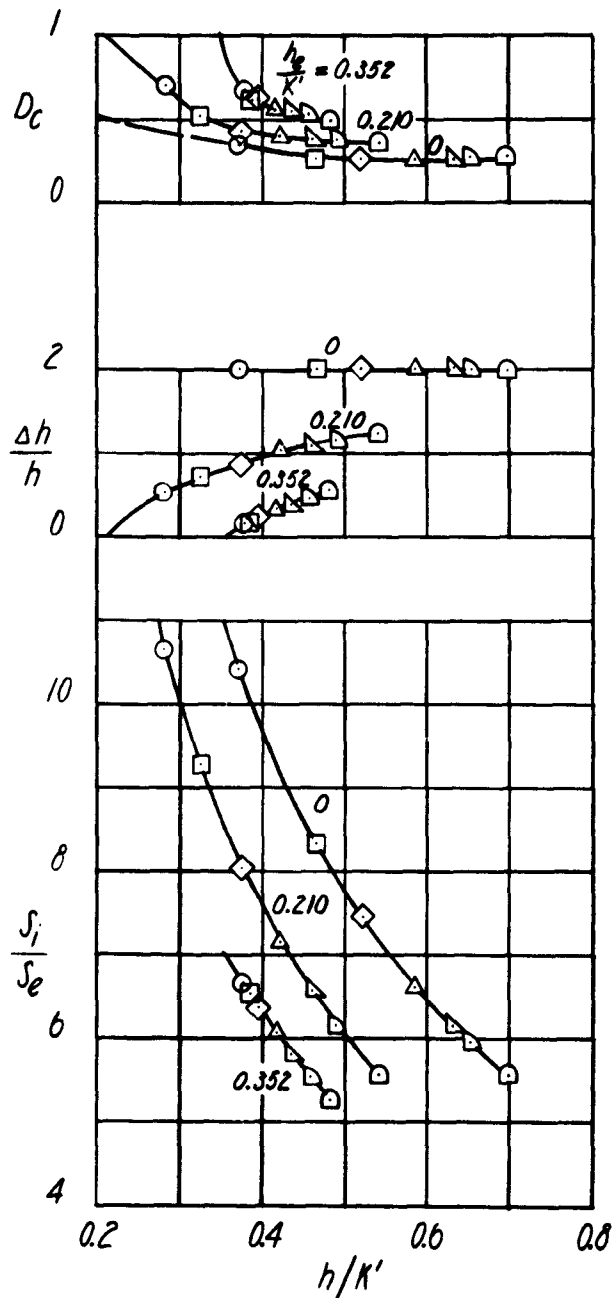


Figure 6 - Variation of the Parameters  $D_c$ ,  $\Delta h/h$ , and  $S_i/S_e$  With  $h/k'$  for Several Values of  $h_e/k'$ .  $\theta = 45^\circ$

(a)  $\frac{S}{S_1} = 5.979$

FIGURE 6a

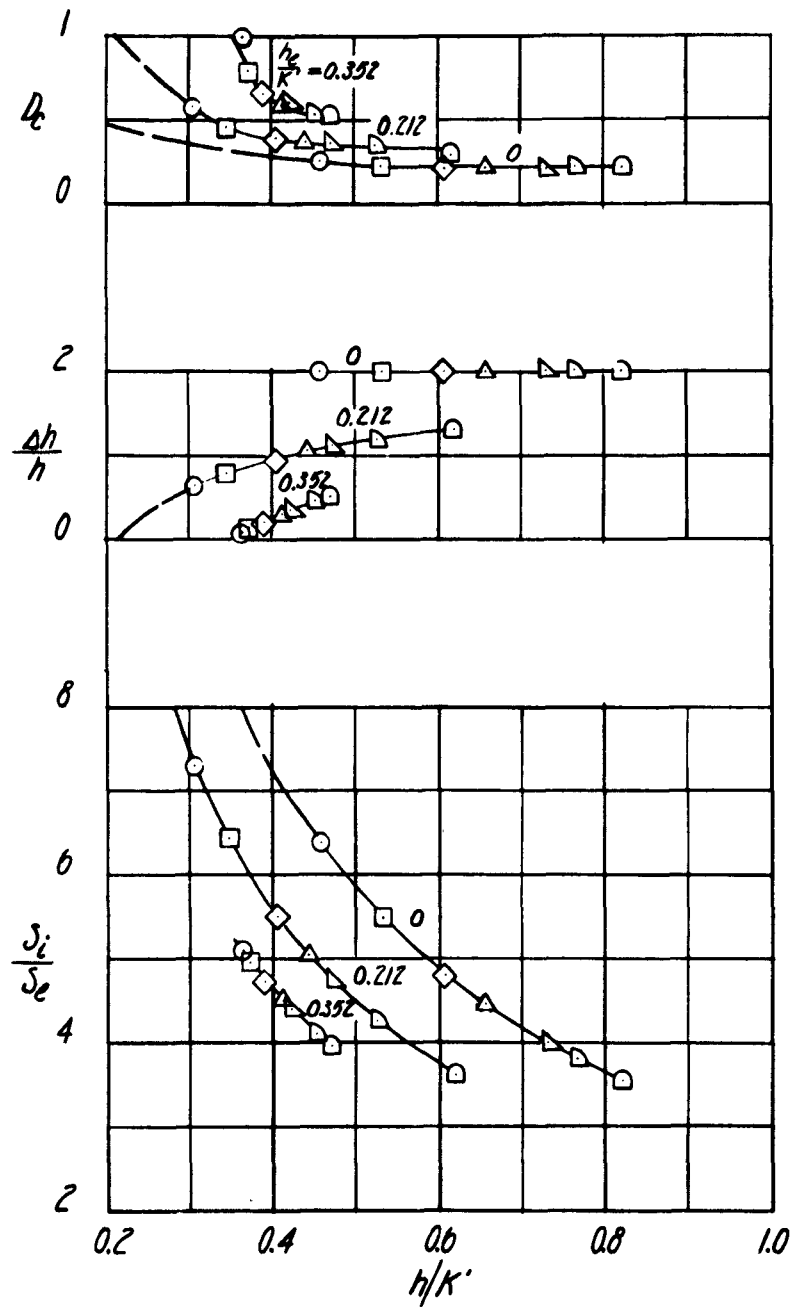


Figure 6 (Continued)

(b)  $\frac{s}{s_1} = 8.123$

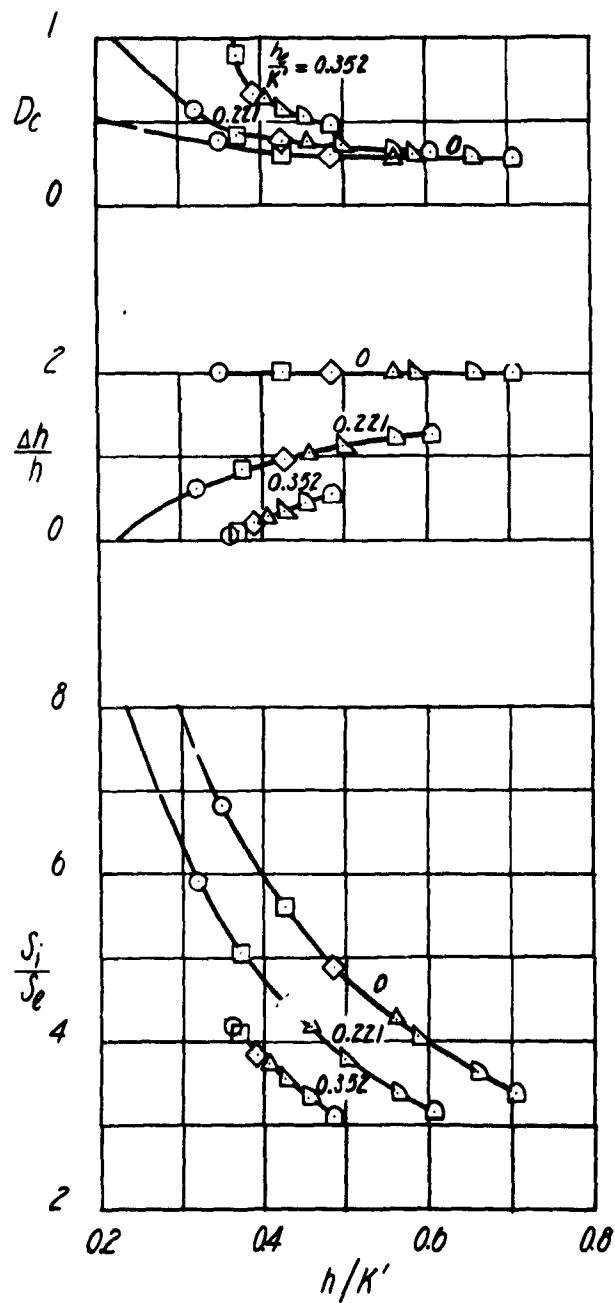


Figure 6 (Continued)

$$(c) \frac{S}{S_1} = 9.980$$

FIGURE 6c

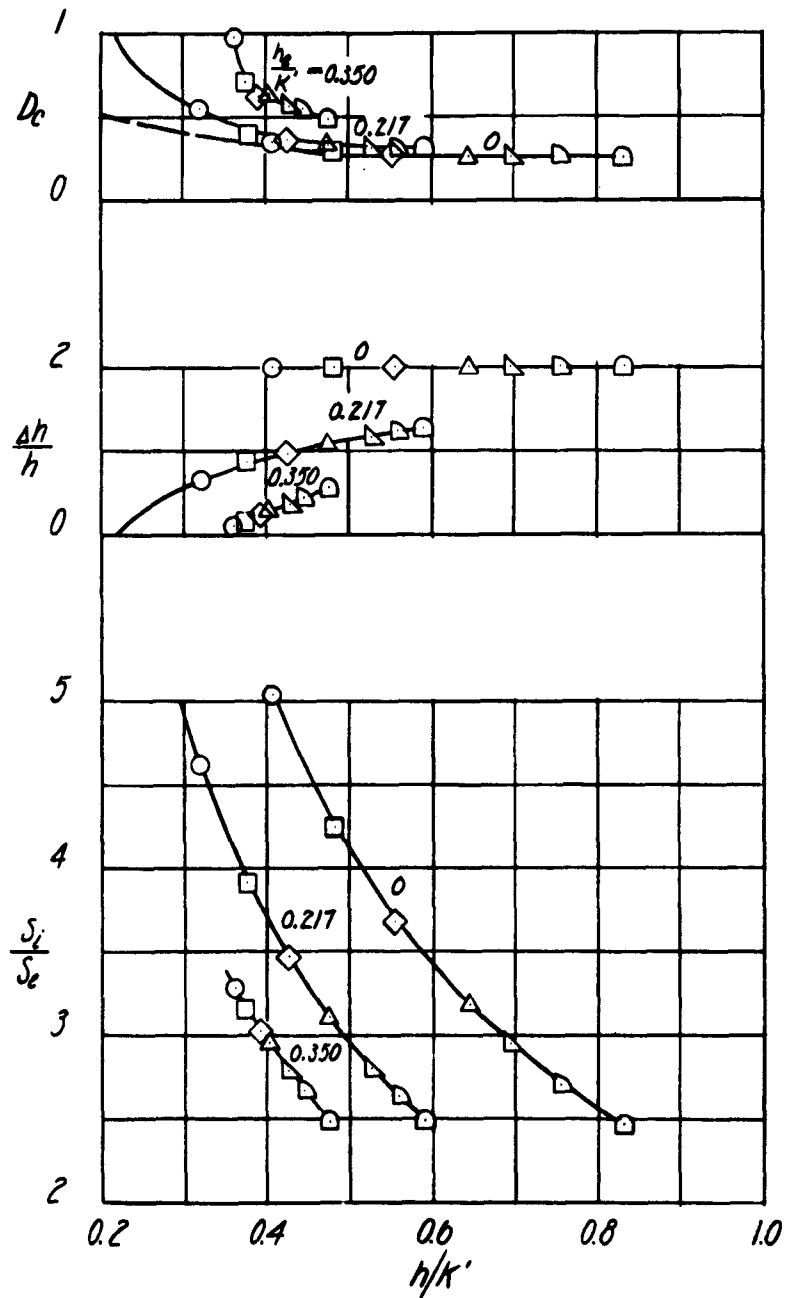


Figure 6 (Continued)

$$(d) \frac{g}{g_1} = 12.583$$

FIGURE 6d

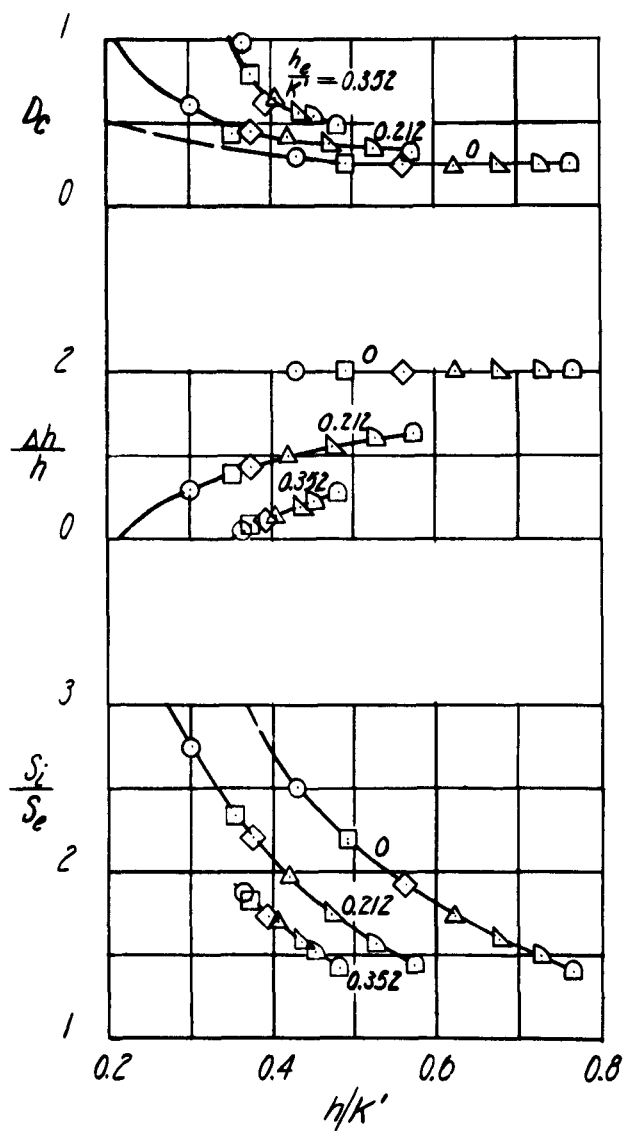


Figure 6 (Continued)

$$(e) \frac{S_2}{S_1} = 17.156$$

FIGURE 6e

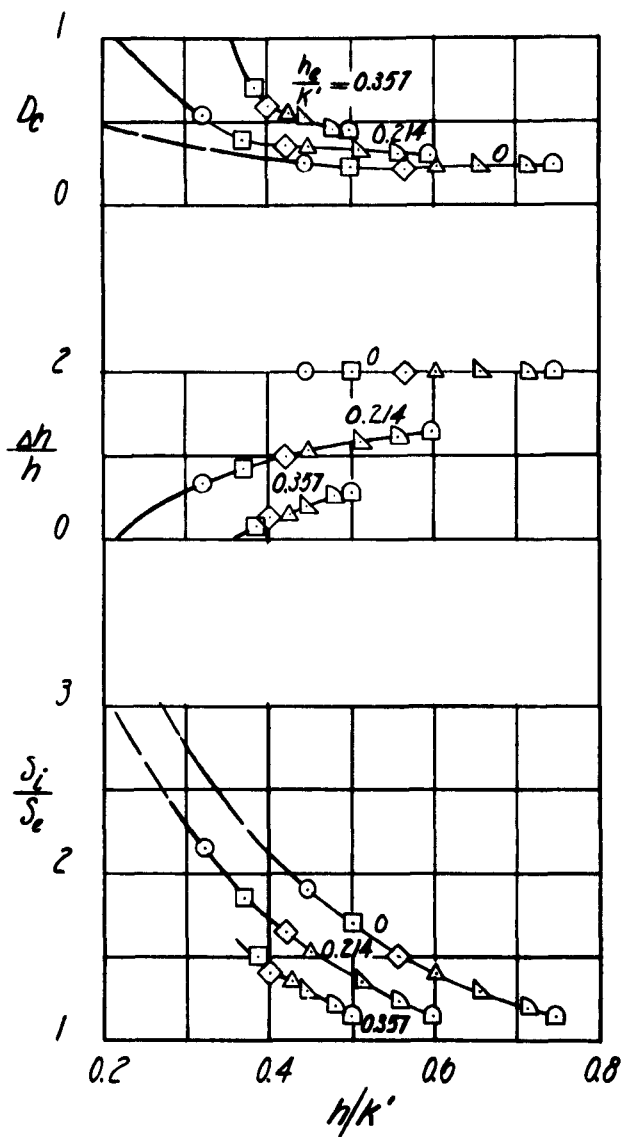


Figure 6 (Concluded)

$$(f) \frac{s}{s_1} = 26.623$$

FIGURE 6f

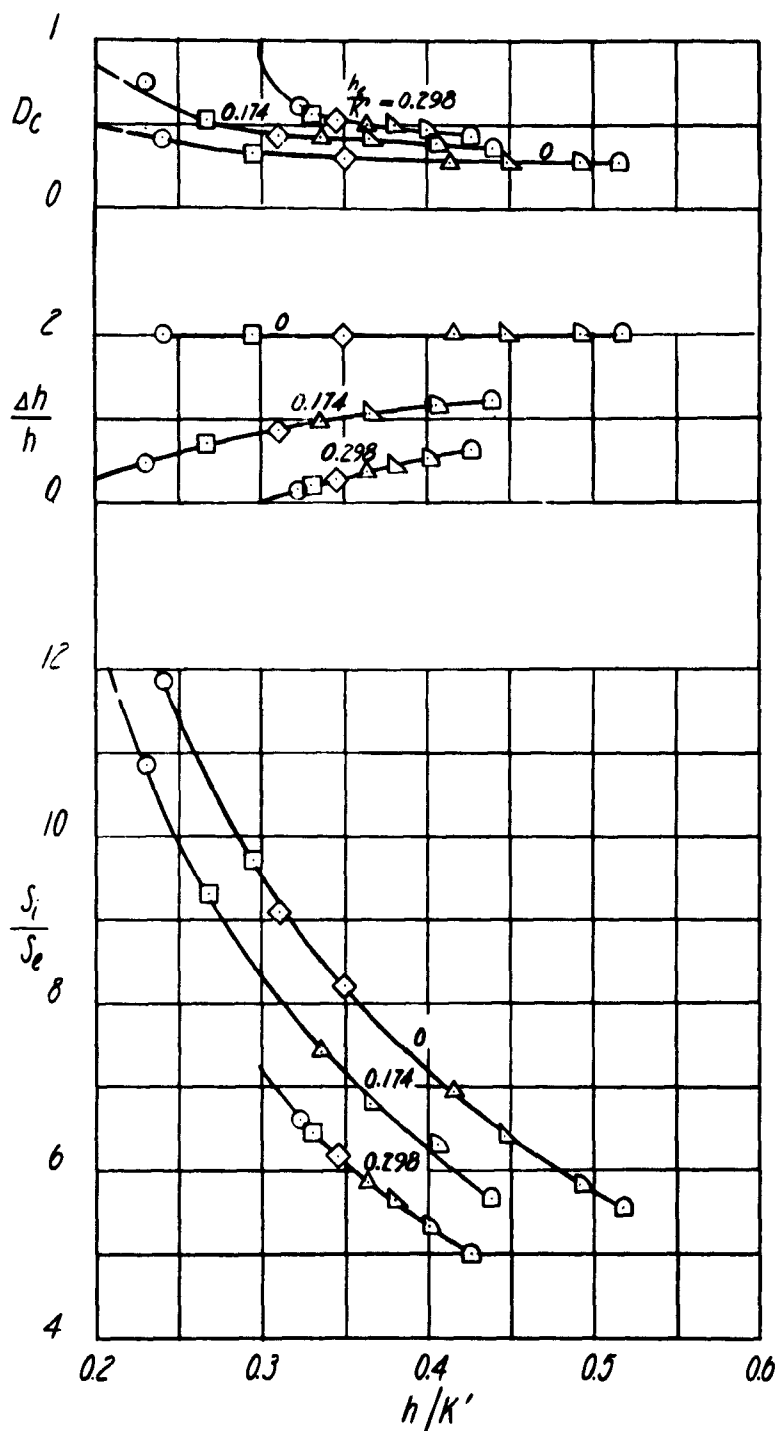


Figure 7 - Variation of the Parameters  $D_c$ ,  $\Delta h/h$ , and  $S_i/S_e$  With  $h/k'$  for Several Values of  $h_e/k'$ .  $\theta = 30^\circ$

(a)  $\frac{S}{S_1} = 5.900$

FIGURE 7a



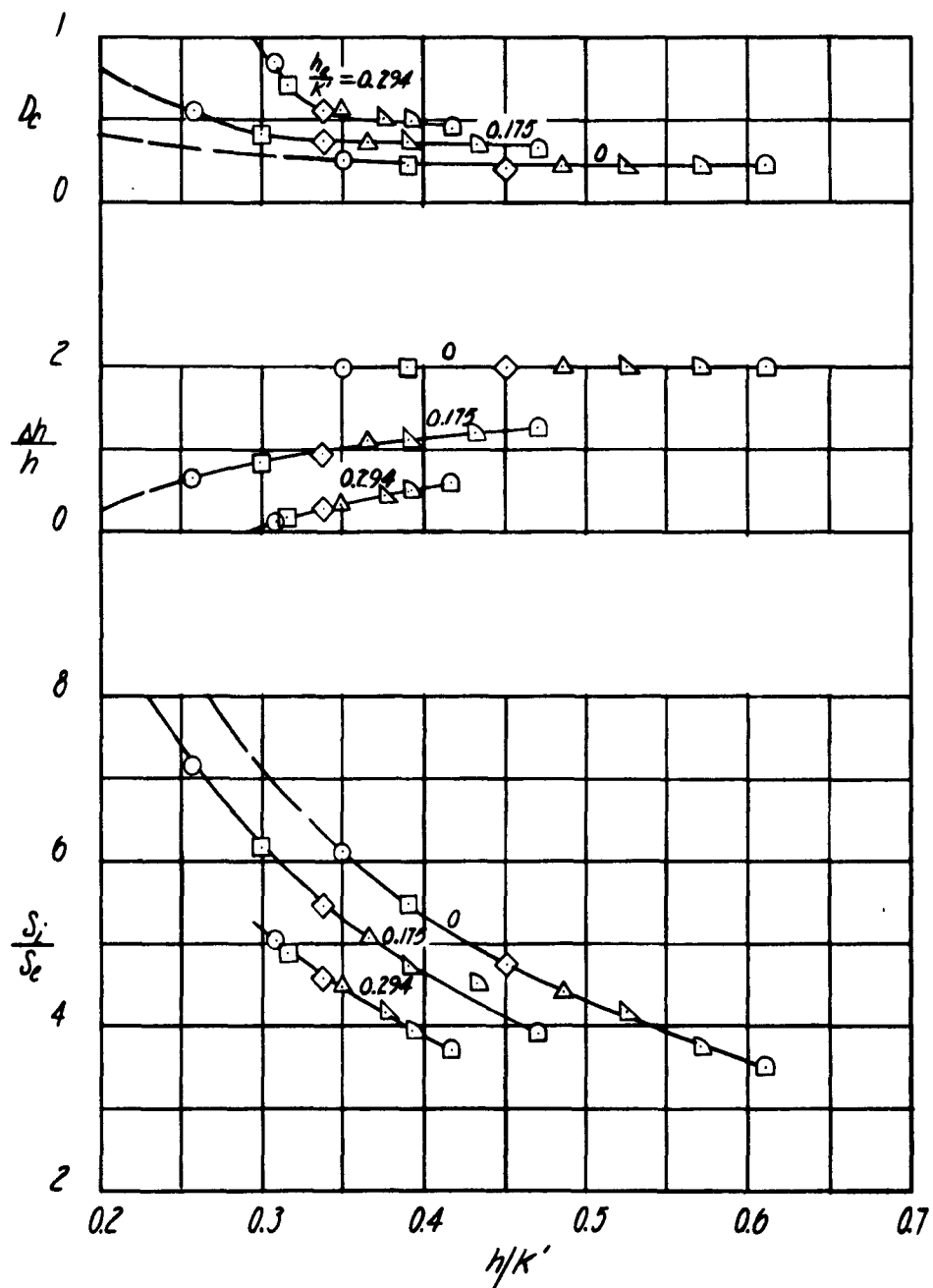


Figure 7 (Continued)

$$(b) \frac{s}{s_1} = 8.028$$

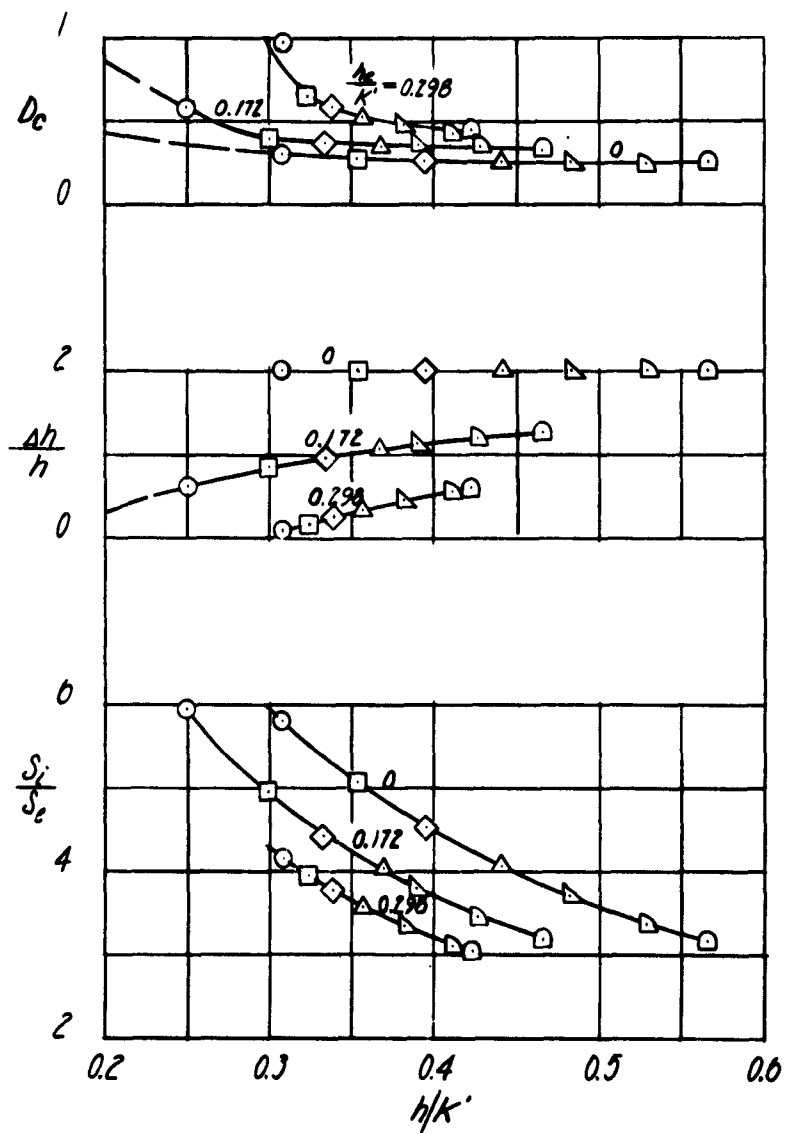


Figure 7 (Continued)

$$(c) \frac{S_2}{S_1} = 9.863$$

FIGURE 7c

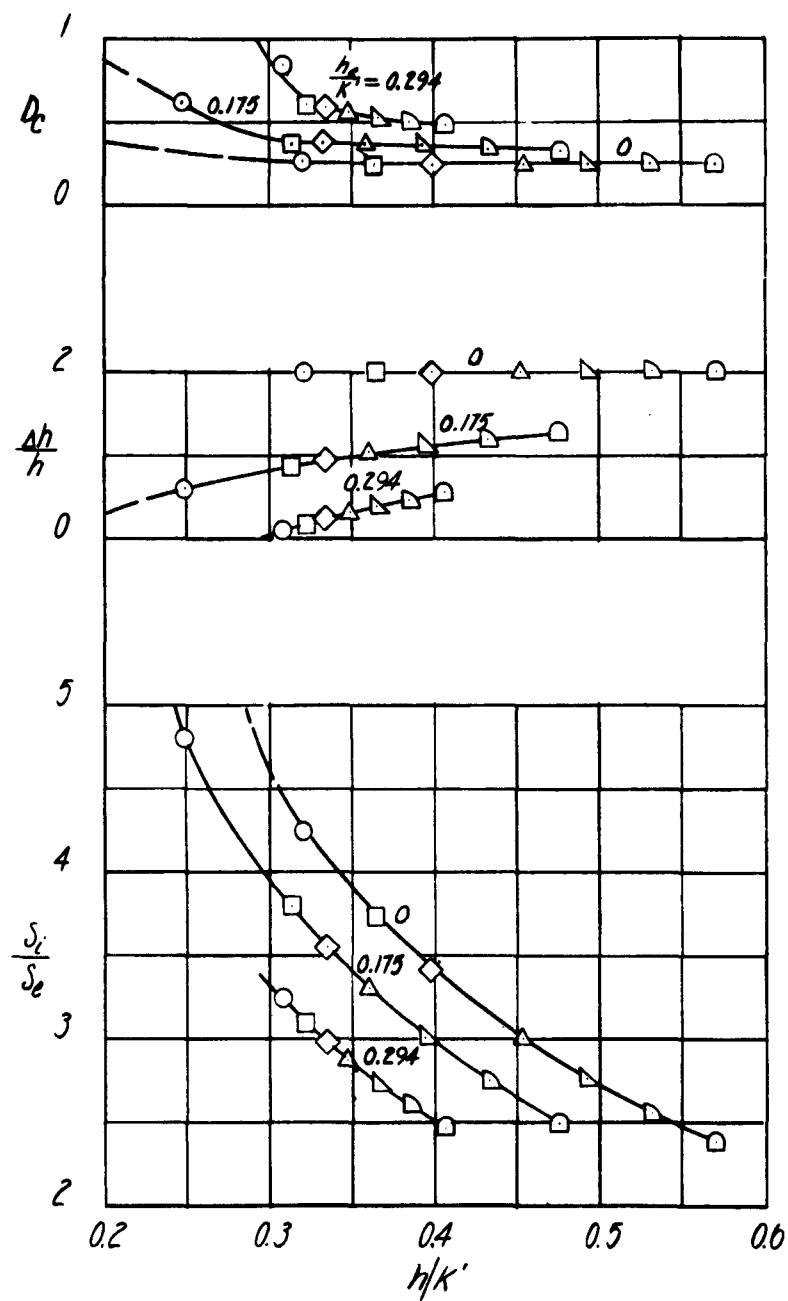


Figure 7 (Continued)

$$(d) \frac{S}{S_1} = 12.435$$

FIGURE 7d

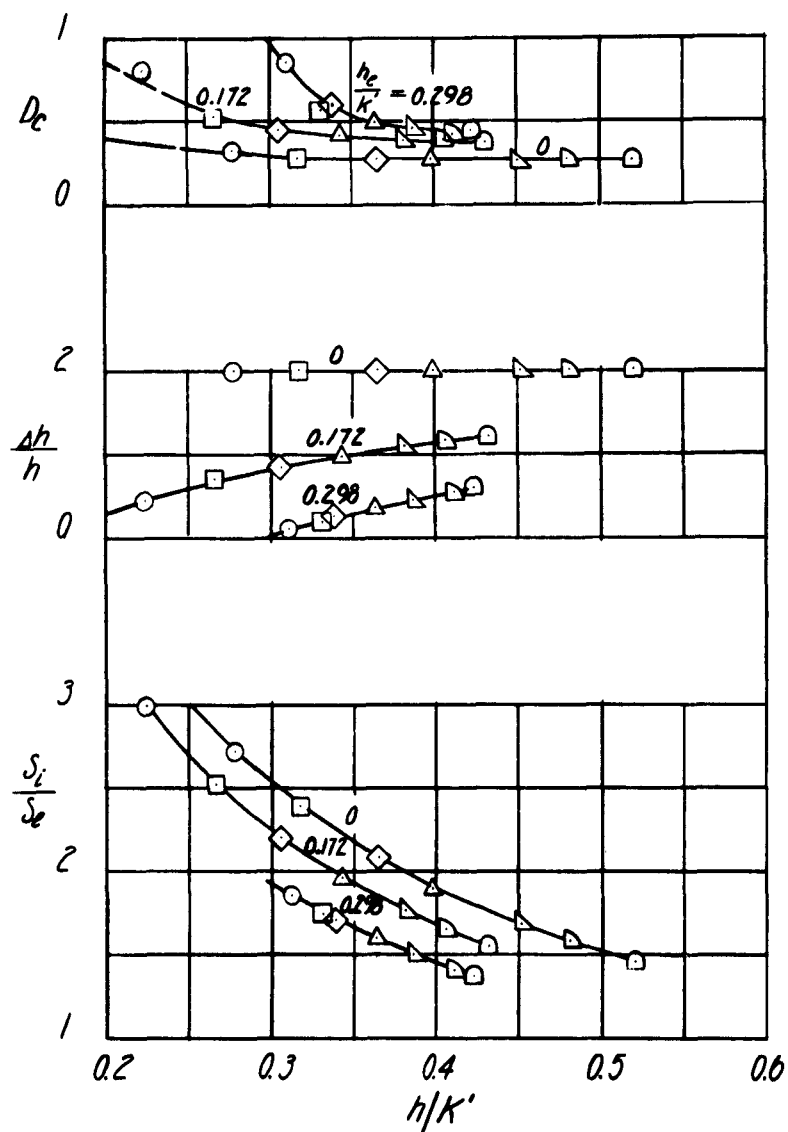


Figure 7 (Continued)

$$(e) \frac{s}{s_1} = 16.957$$

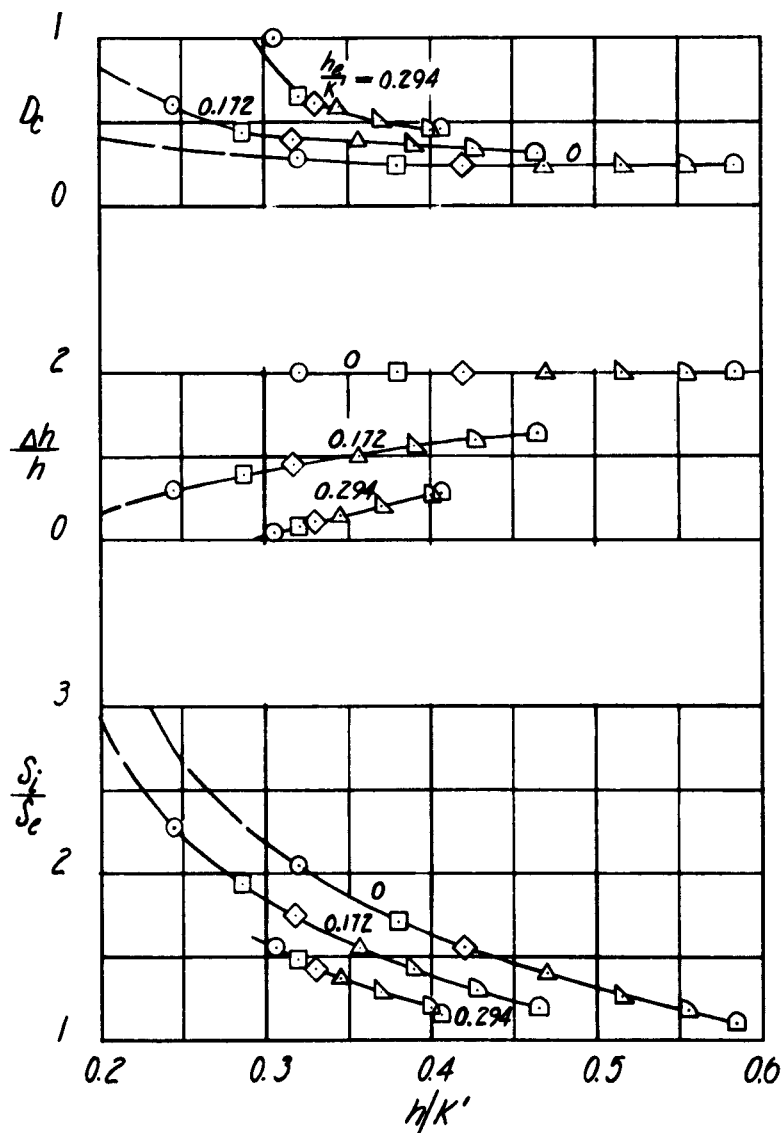


Figure 7 (Concluded)

$$(f) \frac{S}{S_1} = 26.315$$

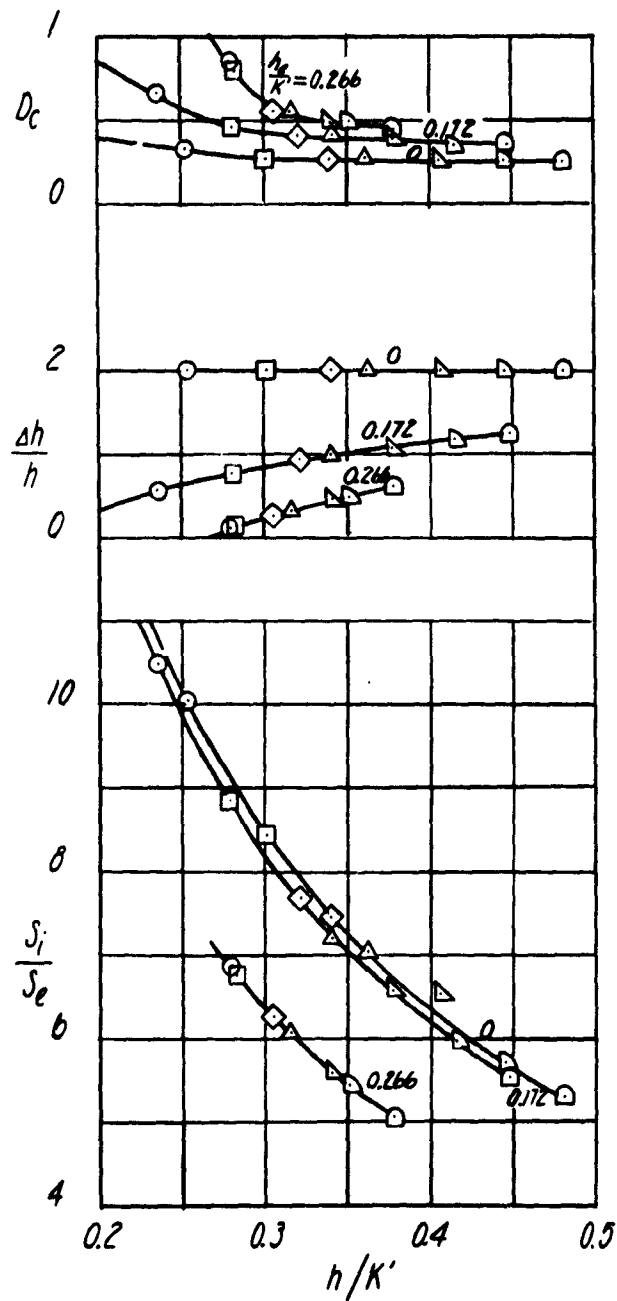


Figure 8 - Variation of the Parameters  $D_c$ ,  $\Delta h/h$ , and  $S_i/S_e$  With  $h/k'$  for Several Values of  $h_e/k'$ .  $\theta = 15^\circ$

(a)  $\frac{S}{S_1} = 5.782$

FIGURE 8a

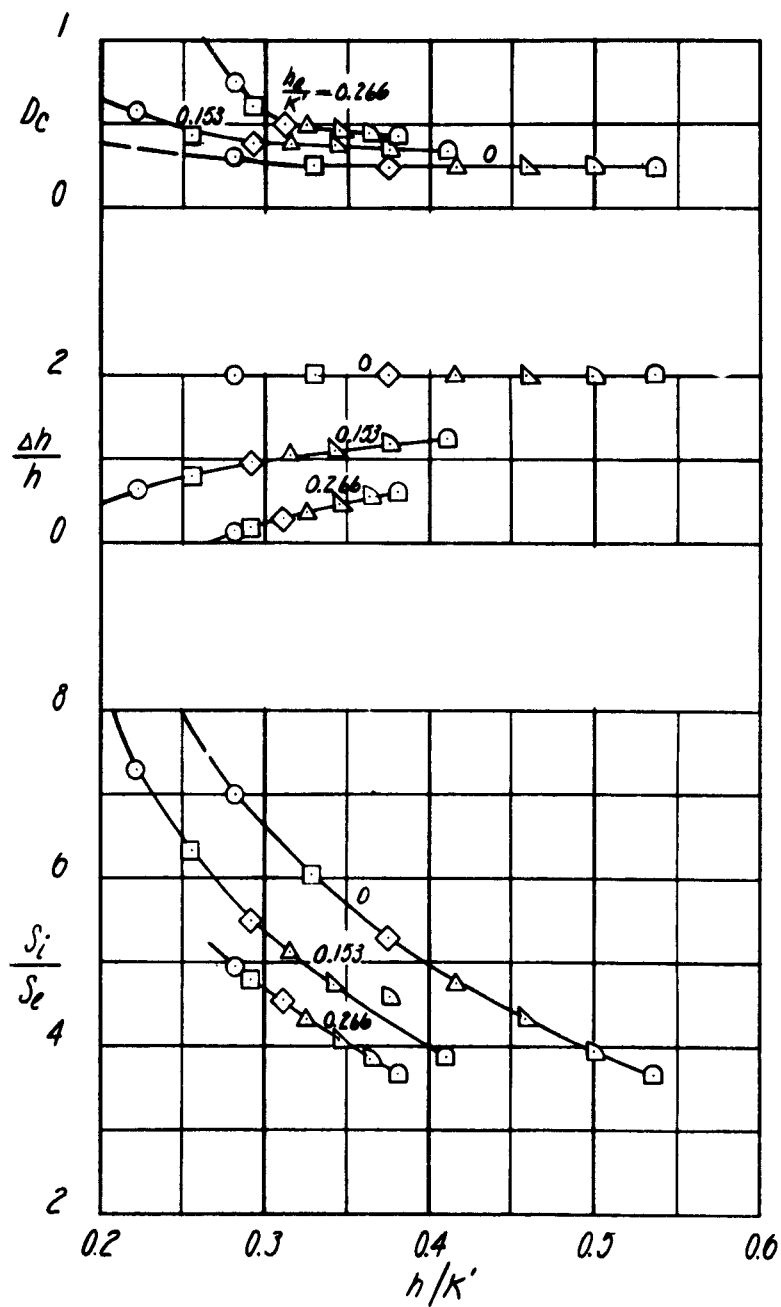


Figure 8 (Continued)

$$(b) \frac{S}{S_1} = 7.868$$

FIGURE 8b

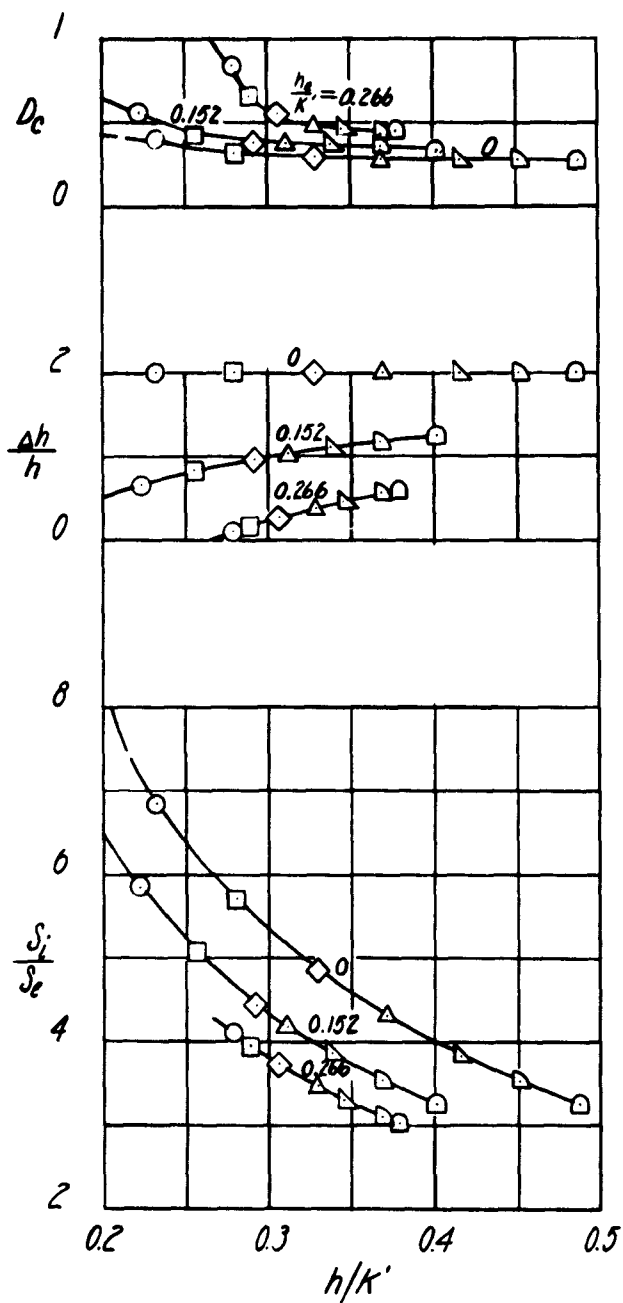


Figure 8 (Continued)

$$(c) \frac{S}{S_i} = 9.666$$

FIGURE 8c



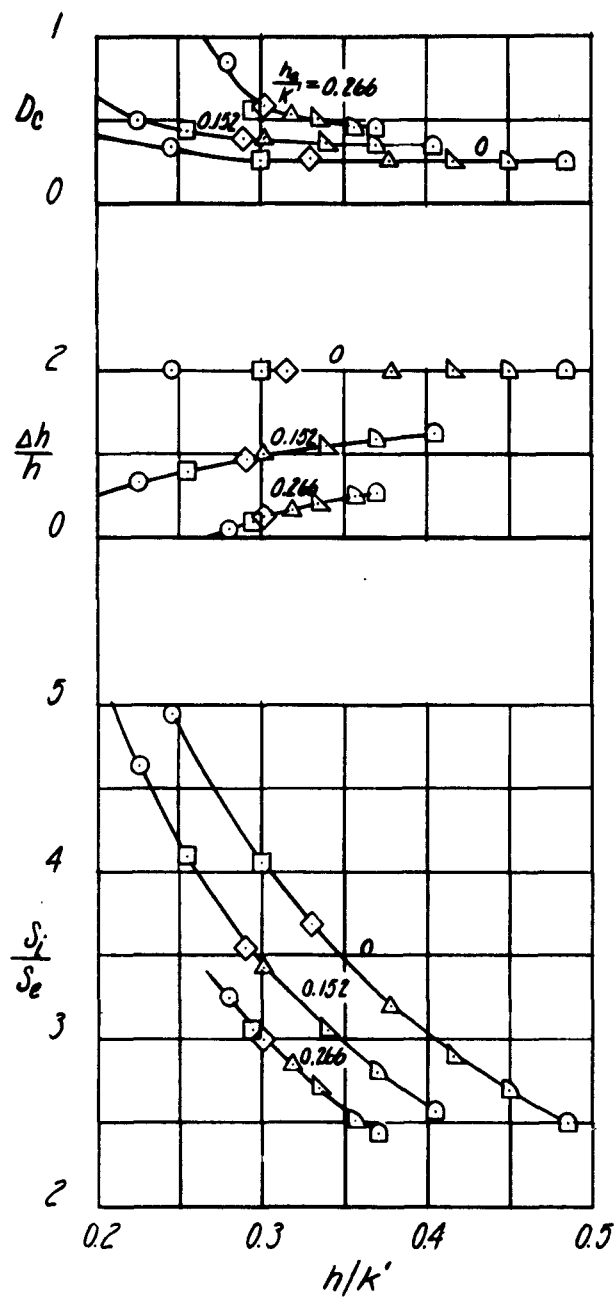


Figure 8 (Continued)

(d)  $\frac{S}{S_1} = 12.187$

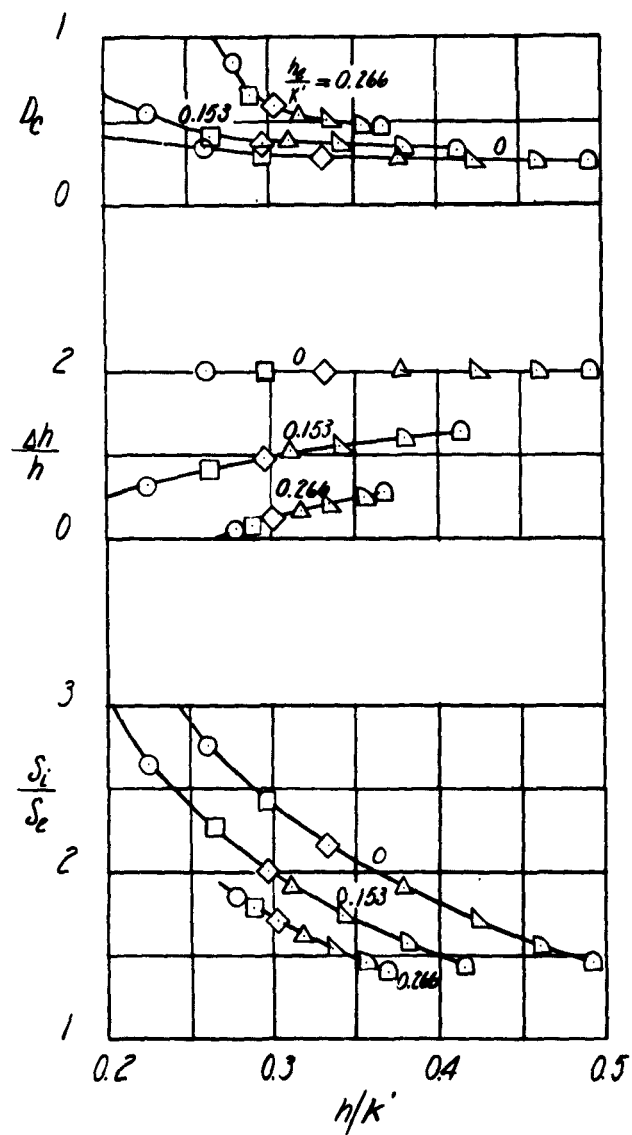


Figure 8 (Continued)

$$(e) \frac{S}{S_1} = 16.619$$

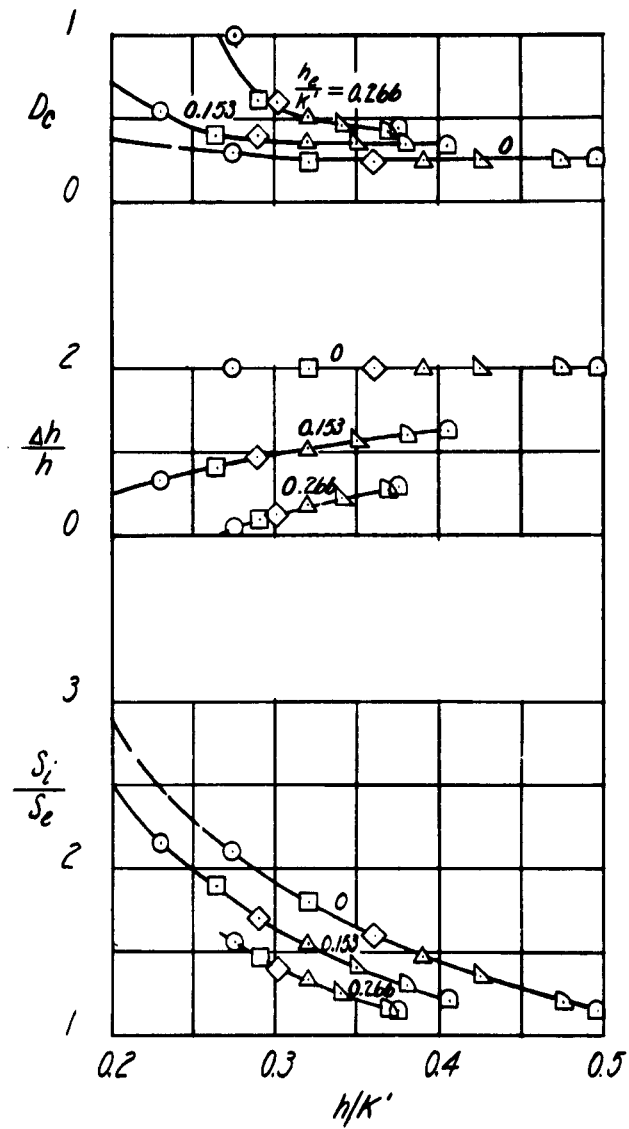


Figure 8 (Concluded)

$$(f) \frac{S}{S_1} = 25.790$$

FIGURE 8f

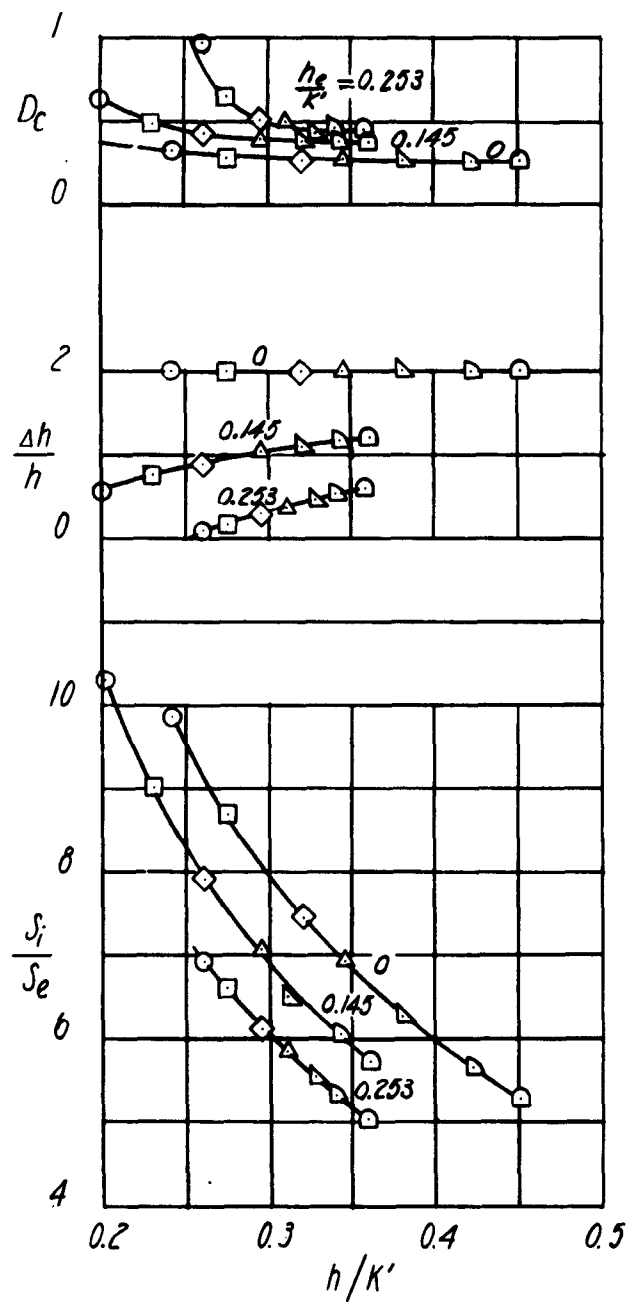


Figure 9 - Variation of the Parameters  $D_c$ ,  $\Delta h/h$ , and  $S_i/S_e$  With  $h/k'$  for Several Values of  $h_e/k'$ .  $\theta = 0^\circ$   
 (a)  $\frac{S}{S_i} = 5.67$

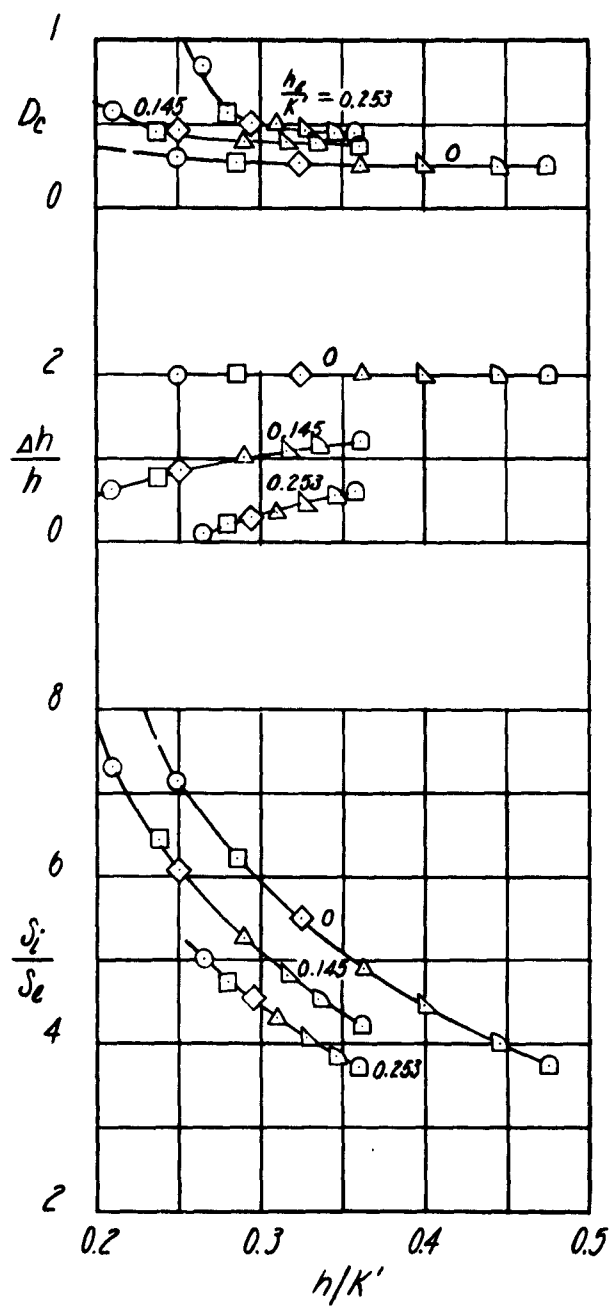


Figure 9 (Continued)

(b)  $\frac{s_2}{s_1} = 7.72$

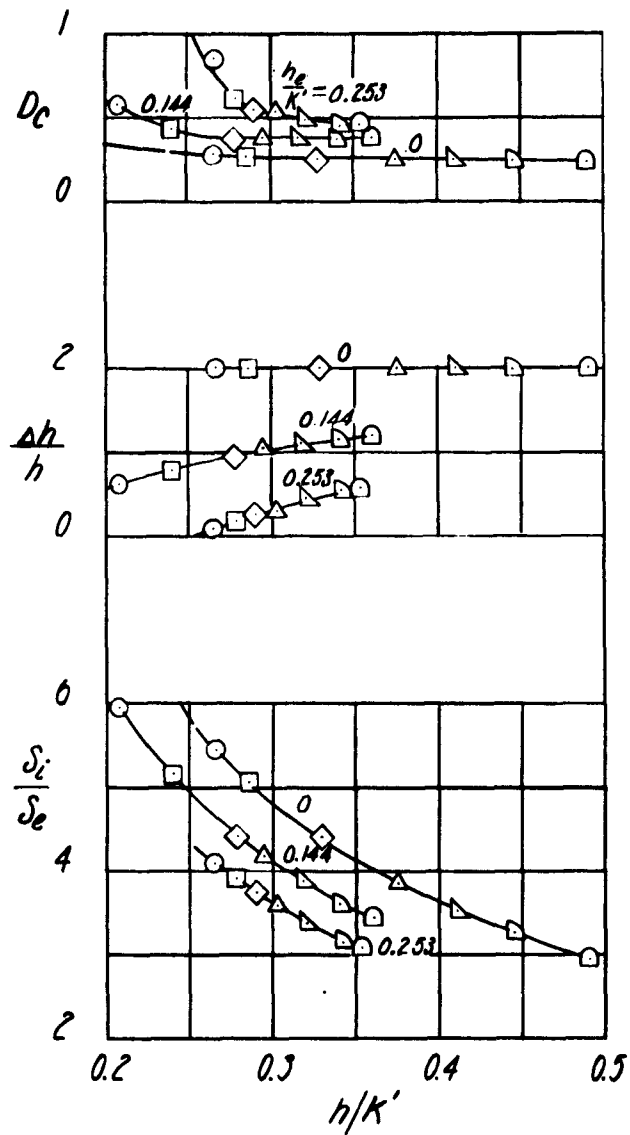


Figure 9 (Continued)

(c)  $\frac{S_2}{S_1} = 9.48$

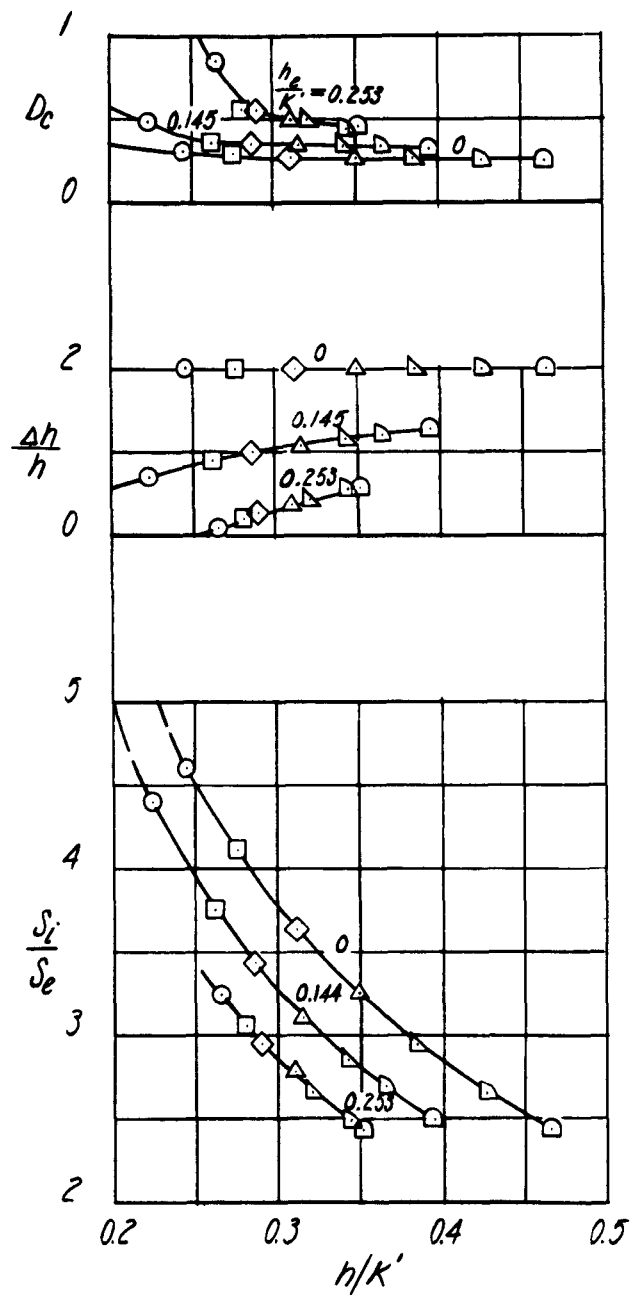


Figure 9 (Continued)

(d)  $\frac{S}{S_1} = 11.96$

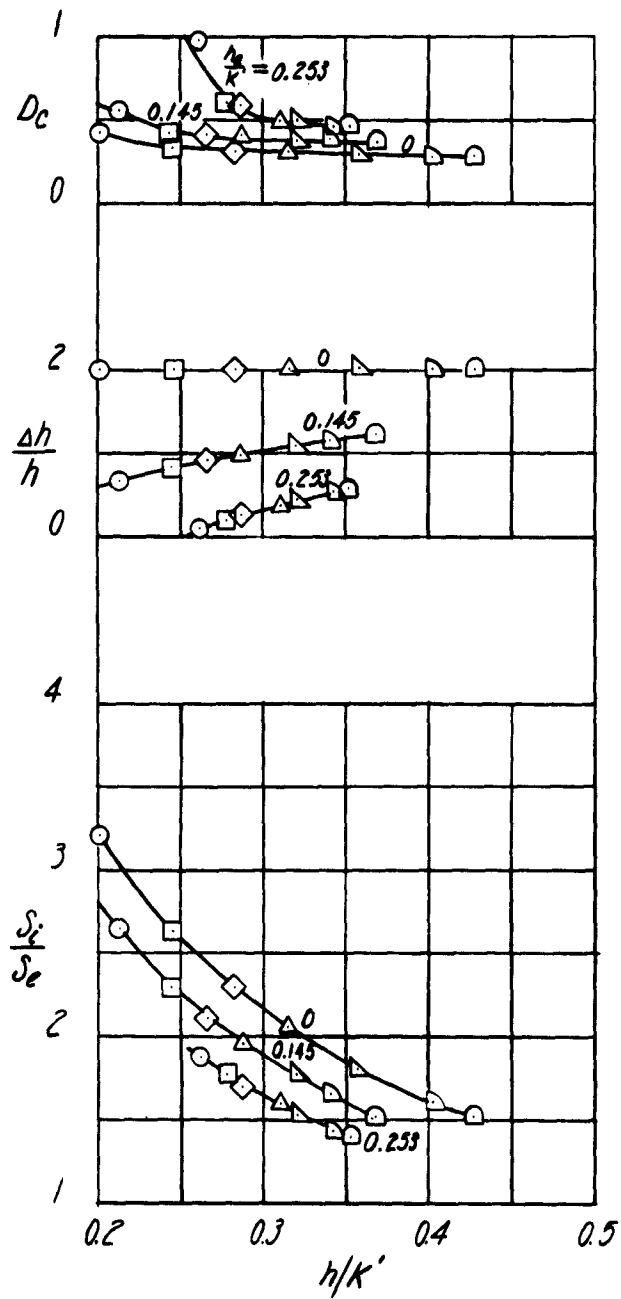


Figure 9 (Continued)

$$(e) \frac{S}{S_1} = 16.31$$

FIGURE 9e



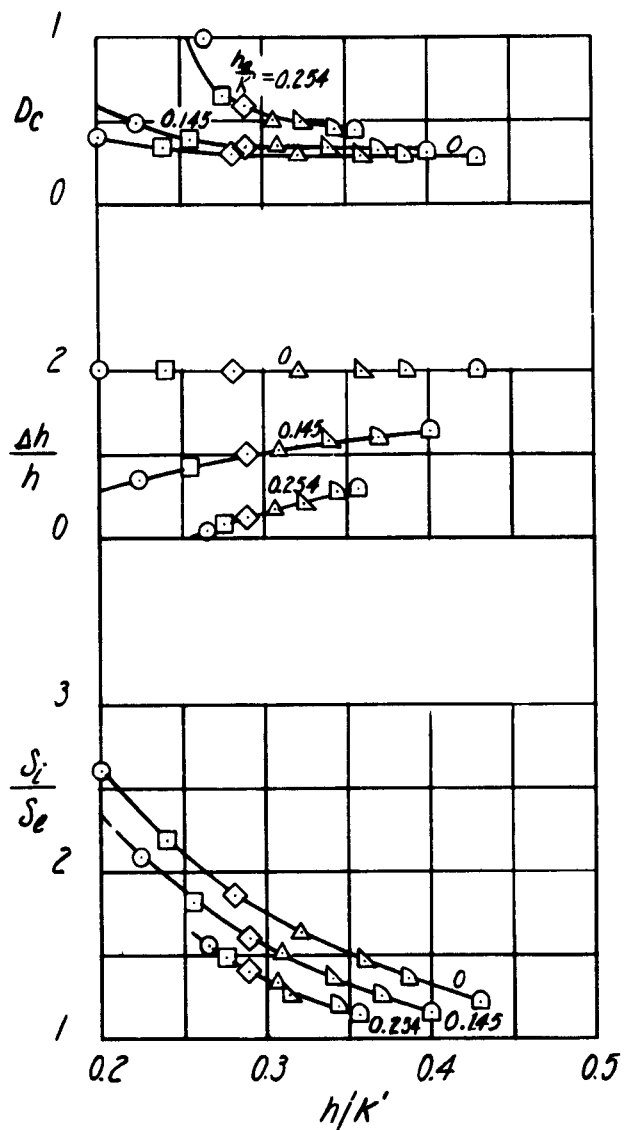


Figure 9 (Concluded)

$$(f) \frac{S}{S_1} = 25.31$$

FIGURE 9f

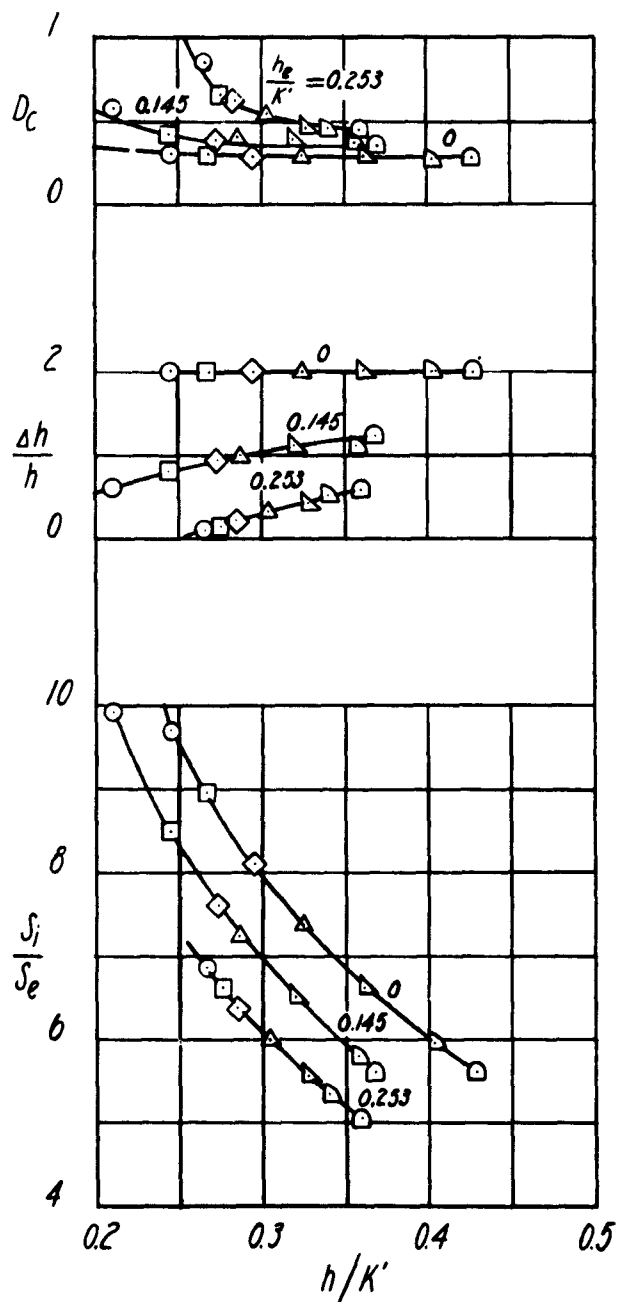


Figure 10 - Variation of the Parameters  $D_c$ ,  $\Delta h/h$ , and  $S_j/S_e$  With  $h/k'$  for Several Values of  $h_e/k'$ .  $\theta = -15^\circ$

(a)  $\frac{S}{S_1} = 5.57$

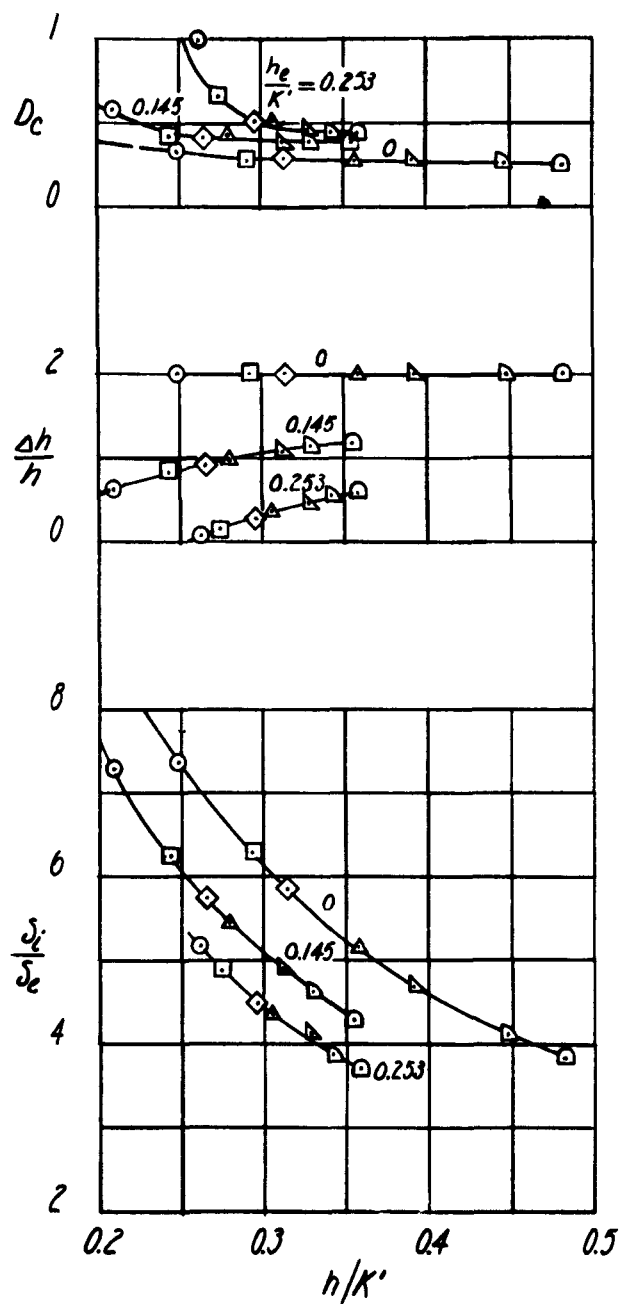


Figure 10 (Continued)

$$(b) \frac{S}{S_1} = 7.57$$

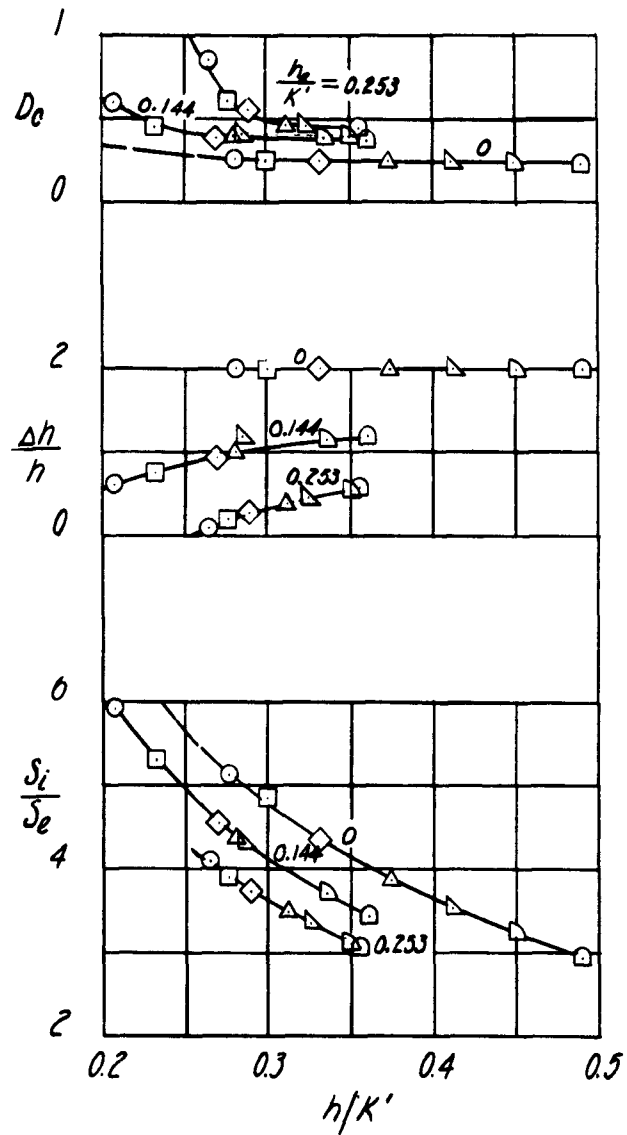


Figure 10 (Continued)

(c)  $\frac{S}{S_1} = 9.30$ 

FIGURE 10c

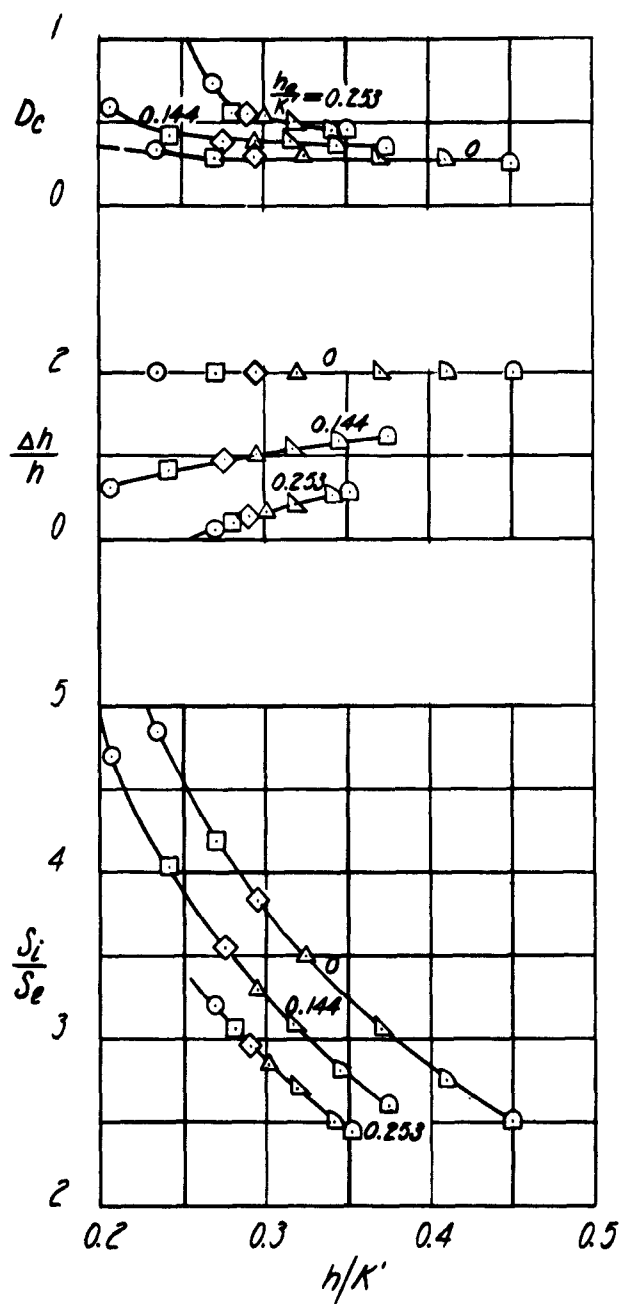


Figure 10 (Continued)

(d)  $\frac{g}{g_1} = 11.73$ 

FIGURE 10d

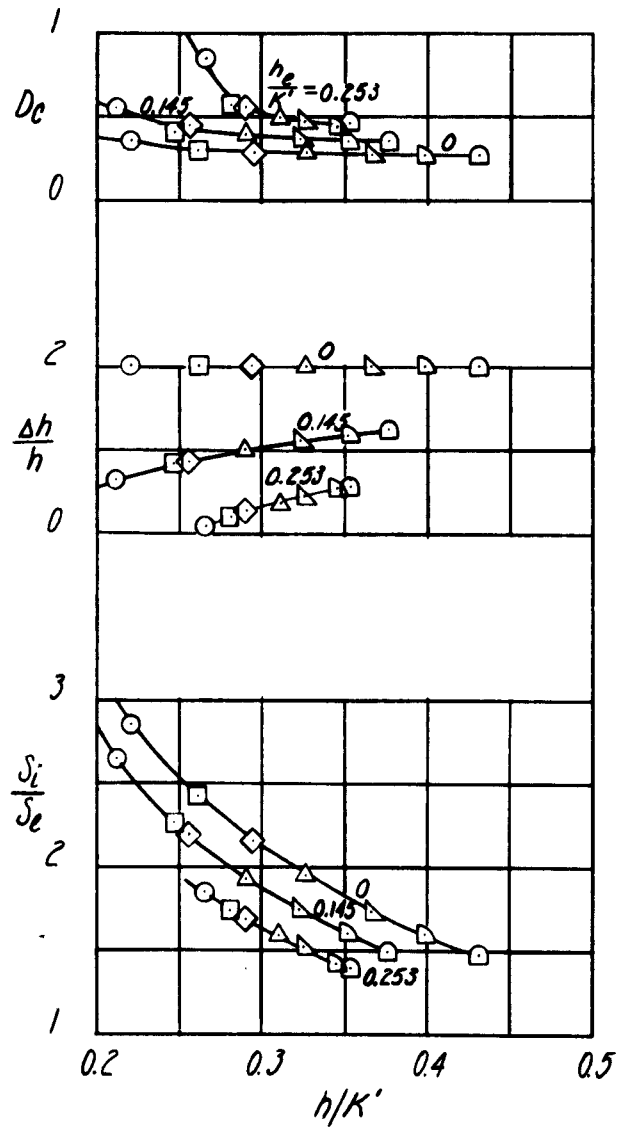


Figure 10 (Continued)

(e)  $\frac{S_2}{S_1} = 16.00$

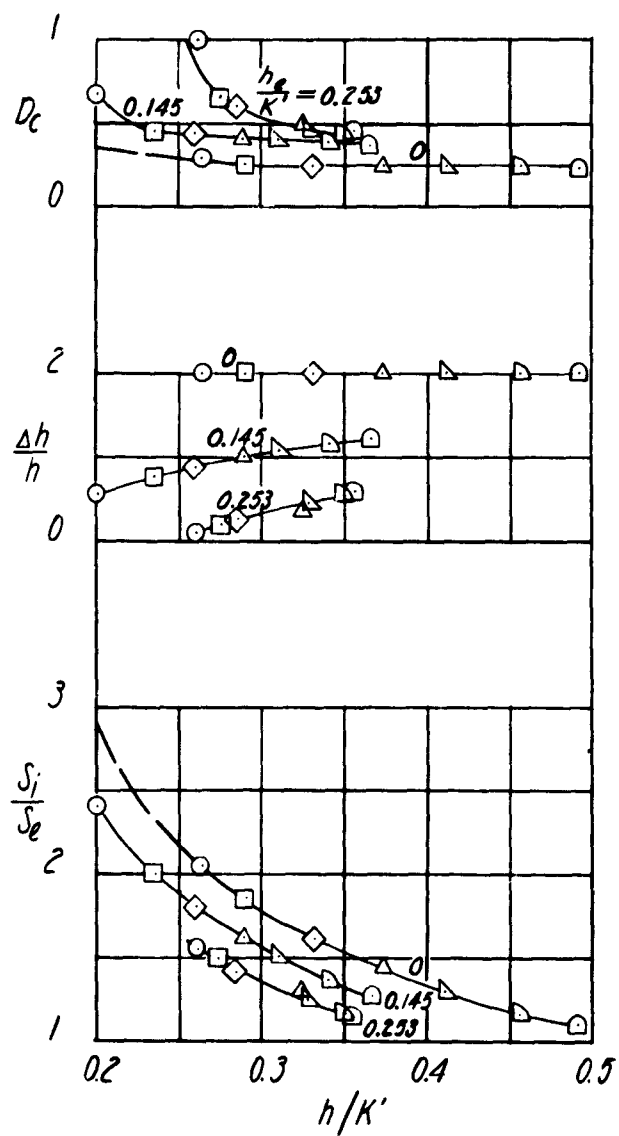


Figure 10 (Concluded)

$$(f) \frac{s}{s_1} = 24.82$$

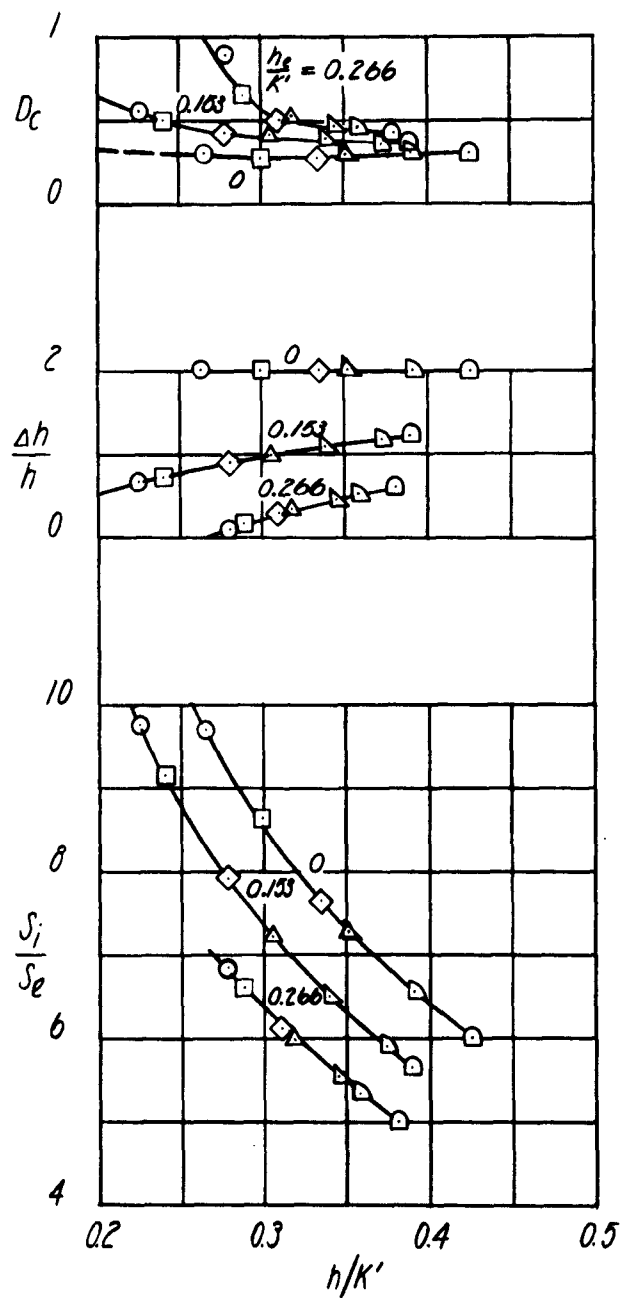


Figure 11 - Variation of the Parameters  $D_c$ ,  $\Delta h/h$ , and  $S_i/S_e$  With  $h/k'$  for Several Values of  $h_e/k'$ .  $\theta = -30^\circ$

(a)  $\frac{S}{S_1} = 5.46$



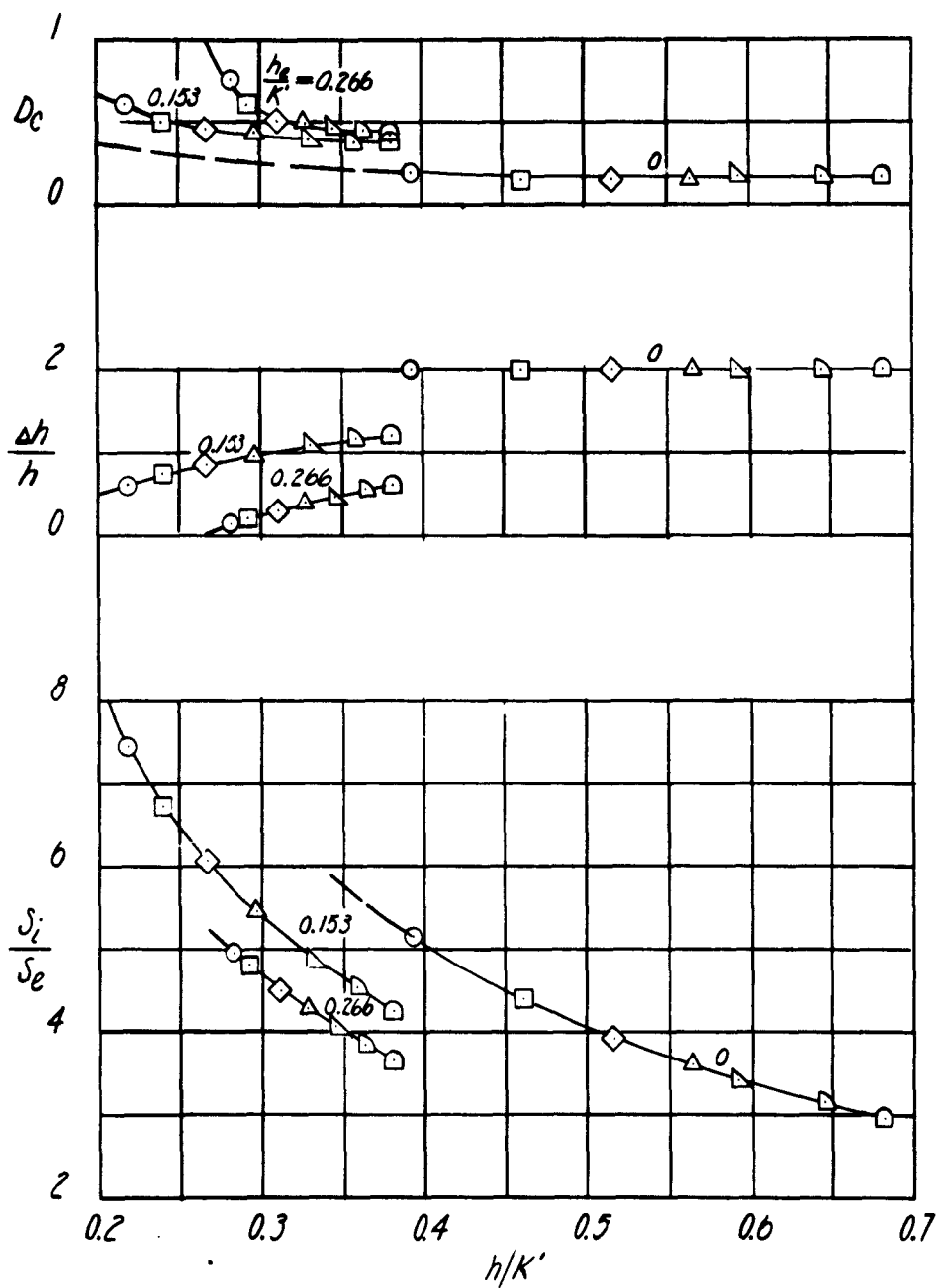


Figure 11 (Continued)

(b)  $\frac{S}{S_1} = 7.44$

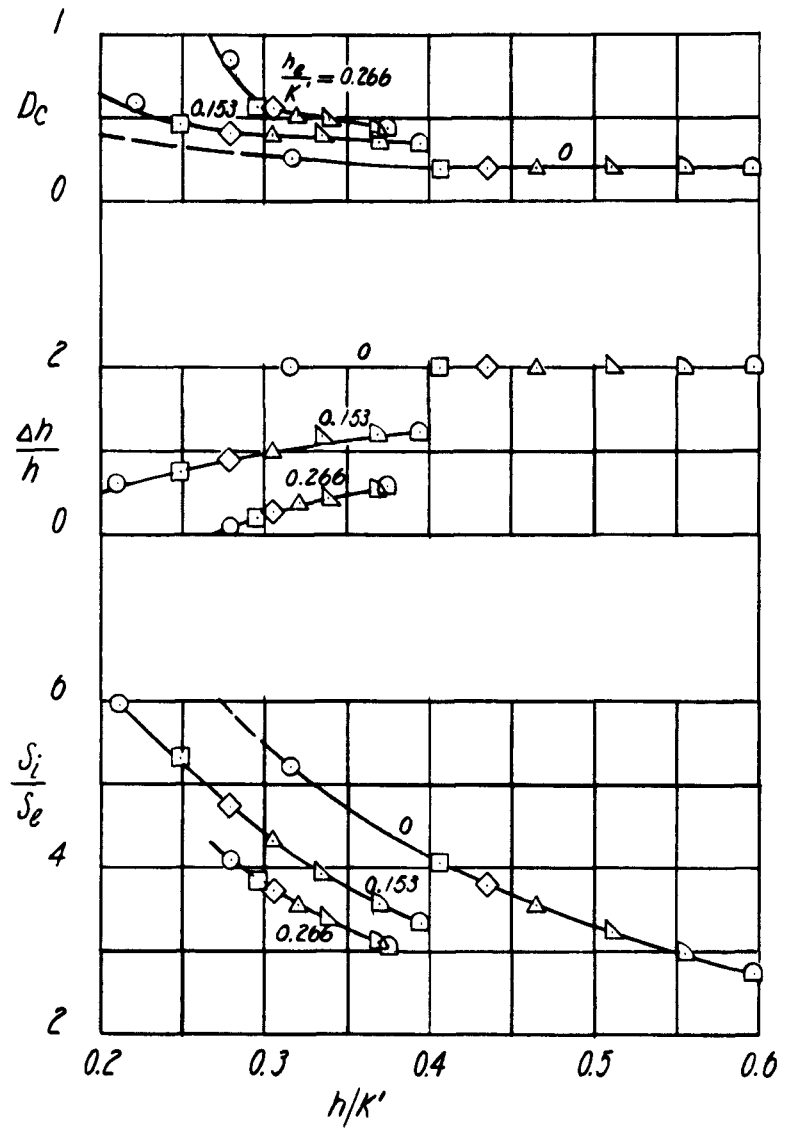


Figure 11 (Continued)

(c)  $\frac{S}{S_1} = 9.14$ 

FIGURE 11c

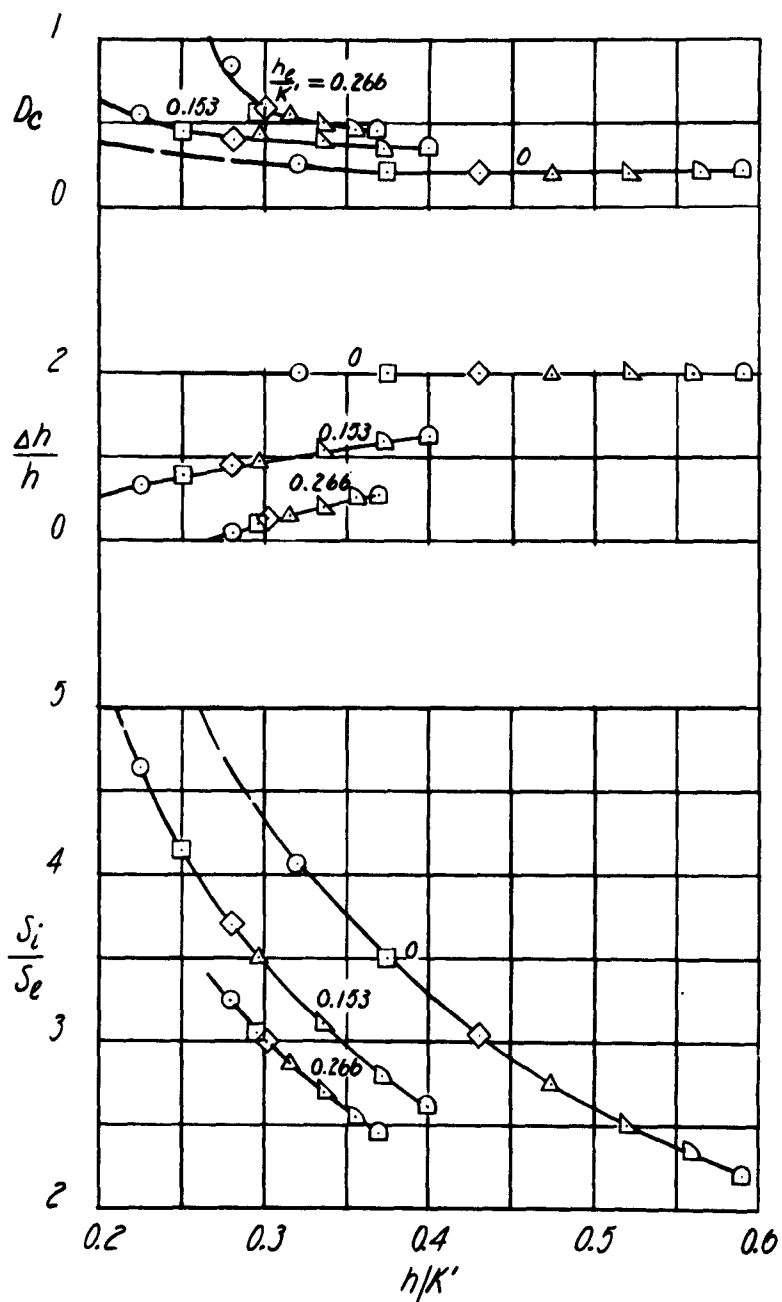


Figure 11 (Continued)

$$(d) \frac{s}{s_1} = 11.52$$

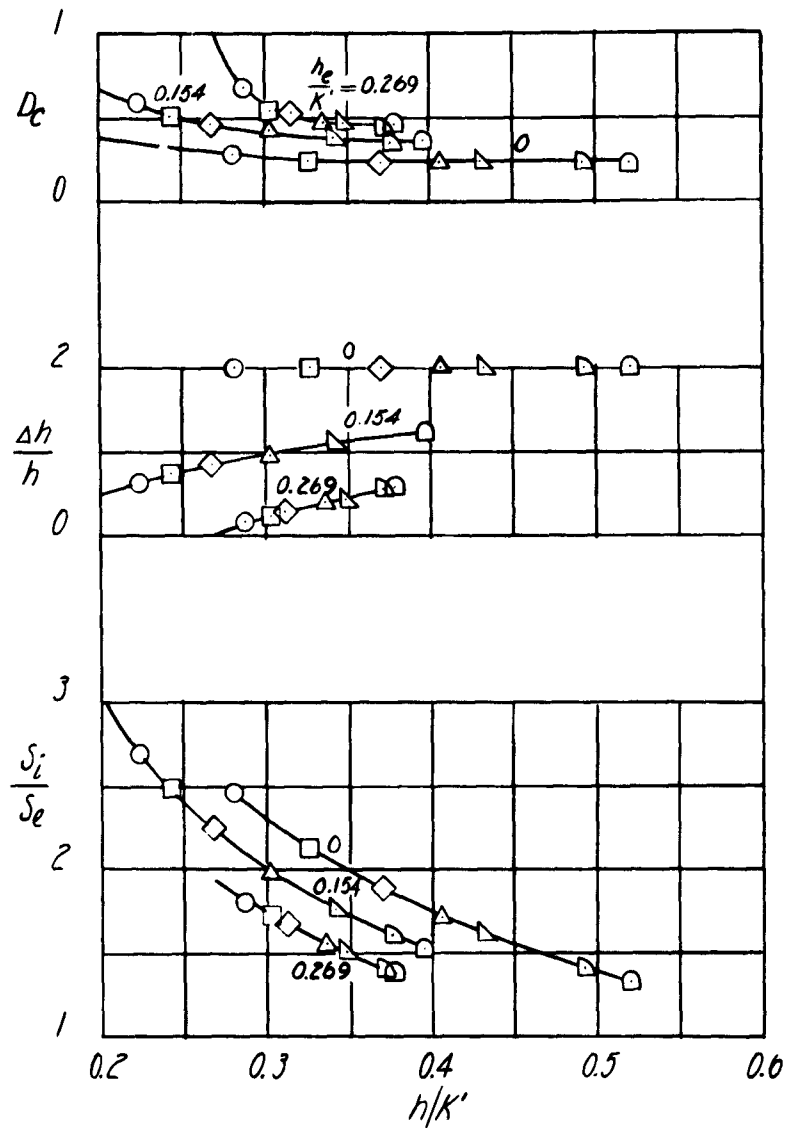


Figure 11 (Continued)

(e)  $\frac{S_i}{S_e} = 15.71$

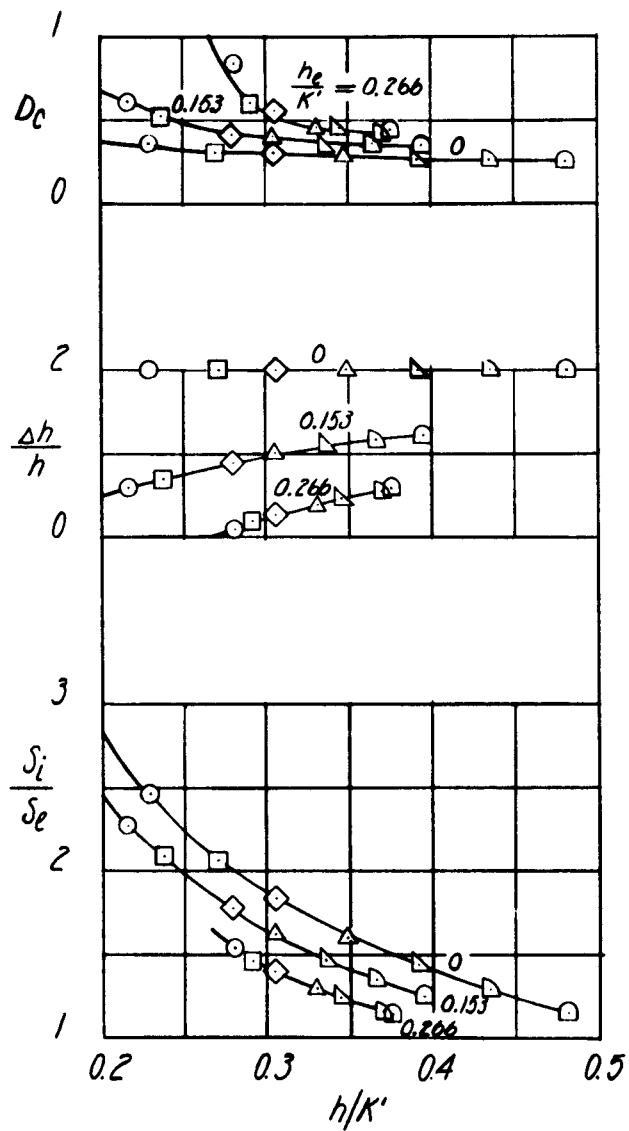


Figure 11 (Concluded)

$$(f) \frac{S}{S_1} = 24.37$$

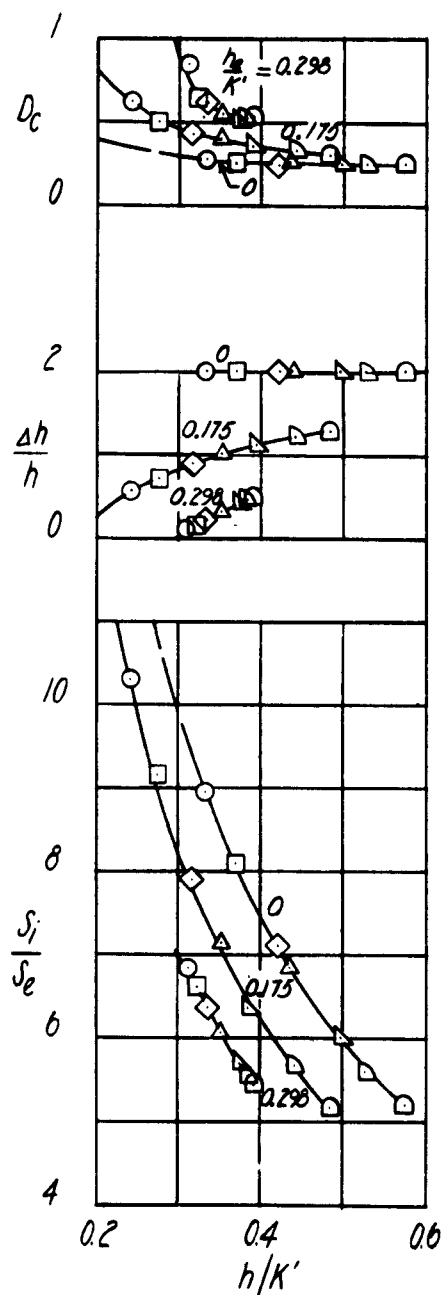


Figure 12 - Variation of the Parameters  $D_c$ ,  $\Delta h/h$ , and  $S_i/S_e$  With  $h/k'$  for Several Values of  $h_e/k'$ .  $\theta = -45^\circ$   
 (a)  $\frac{S}{S_1} = 5.38$

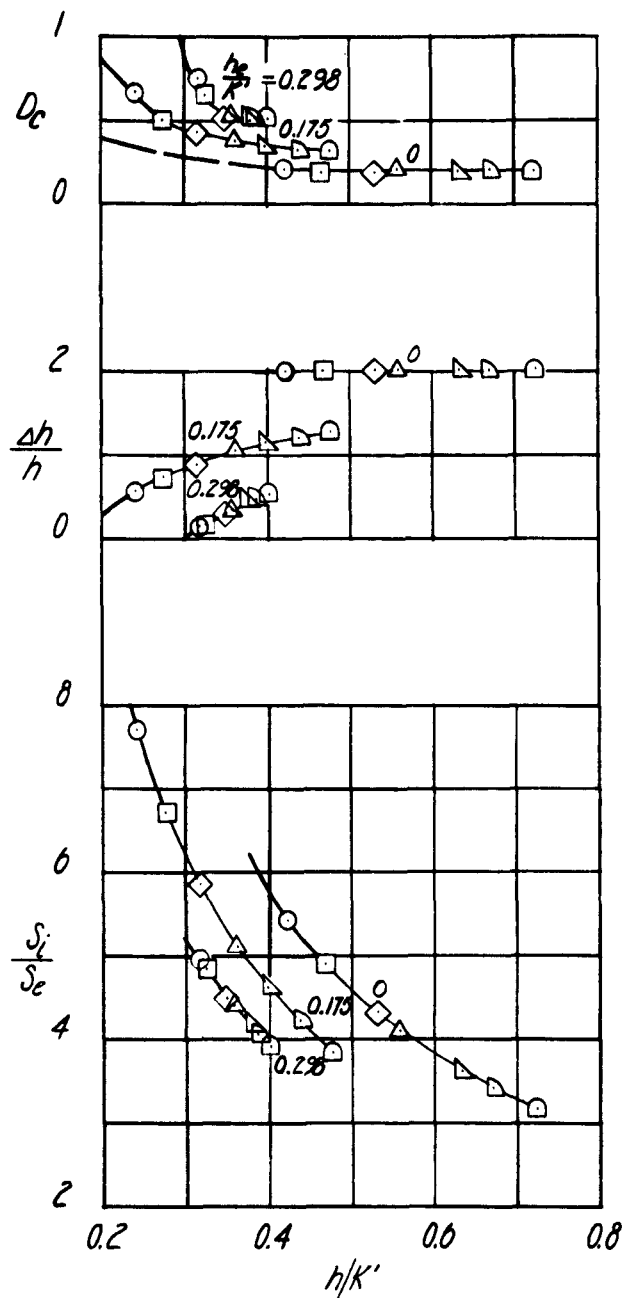


Figure 12 (Continued)

(b)  $\frac{S_j}{S_1} = 7.32$

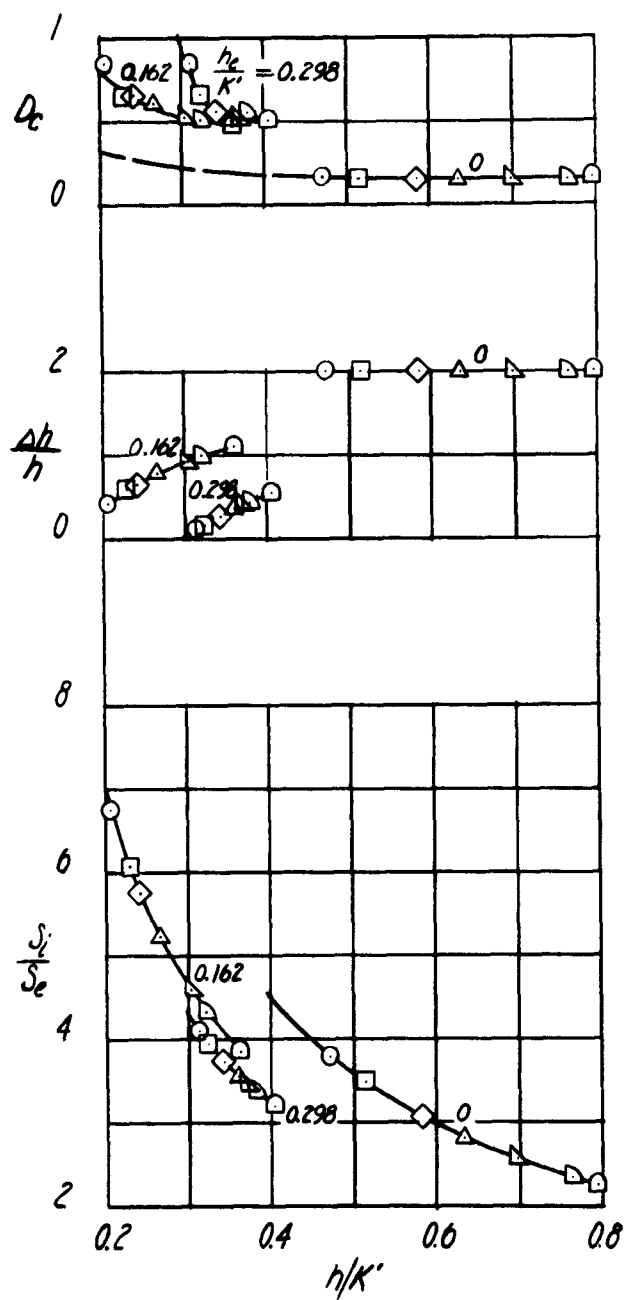


Figure 12 (Continued)

(c)  $\frac{S}{S_1} = 8.99$ 

FIGURE 12c



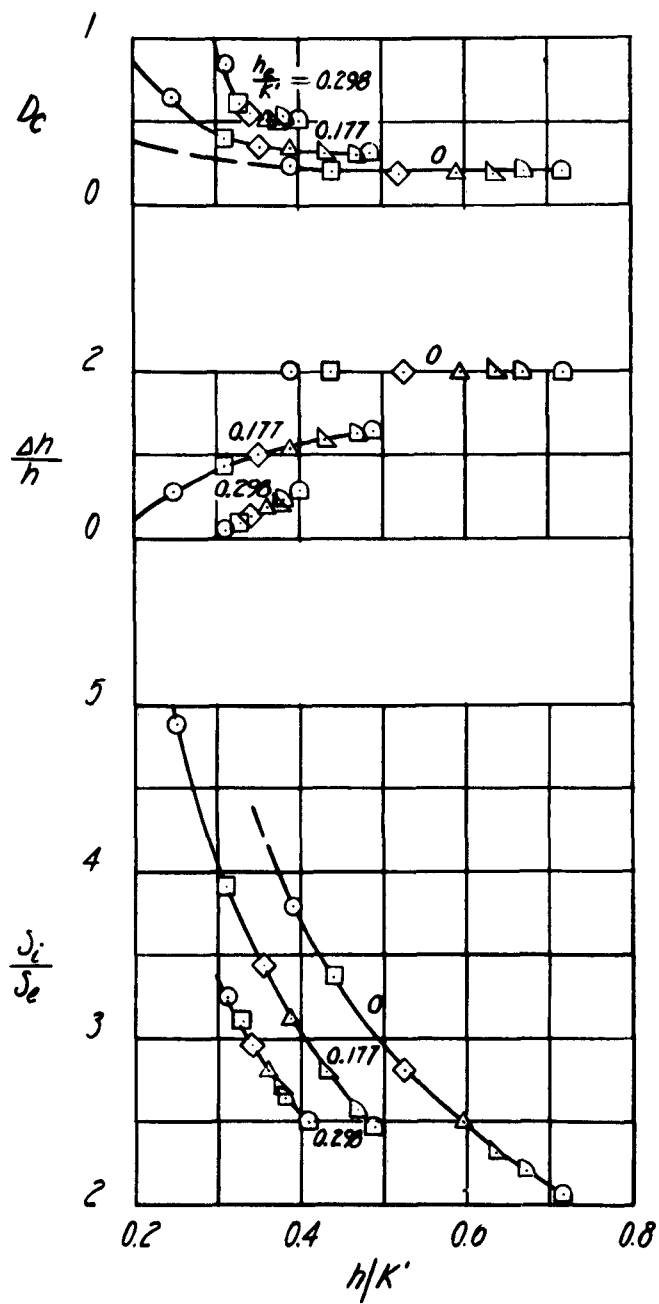


Figure 12 (Continued)

(d)  $\frac{S}{S_1} = 11.34$ 

FIGURE 12d

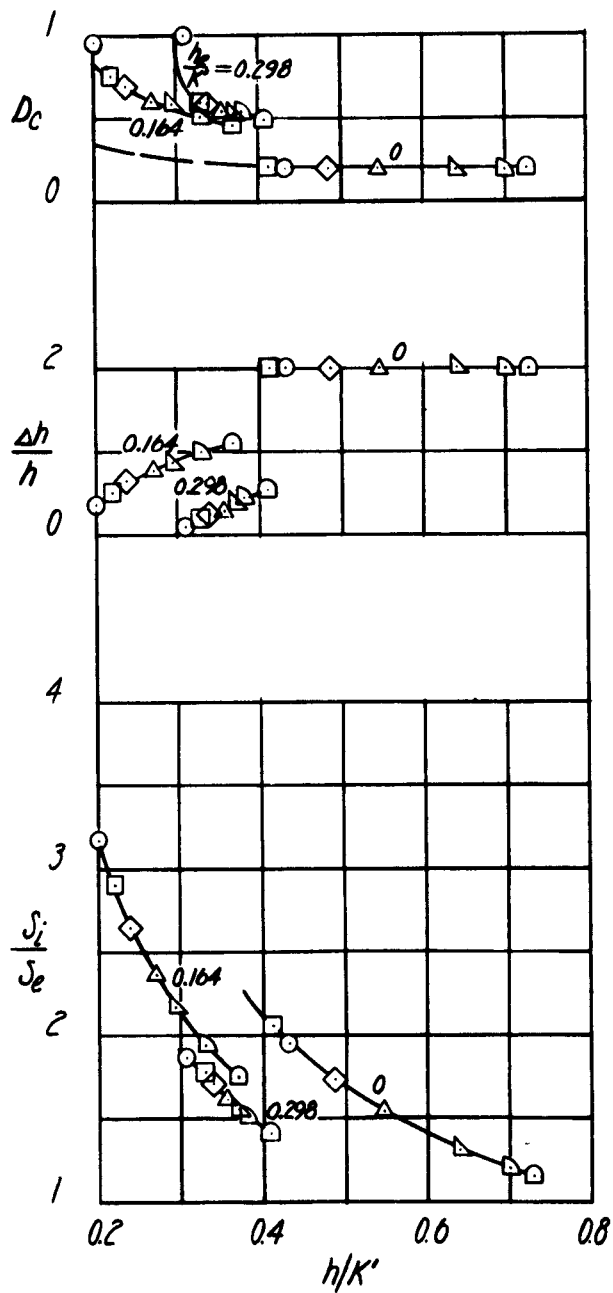


Figure 12 (Continued)

$$(e) \frac{s}{s_1} = 15.46$$

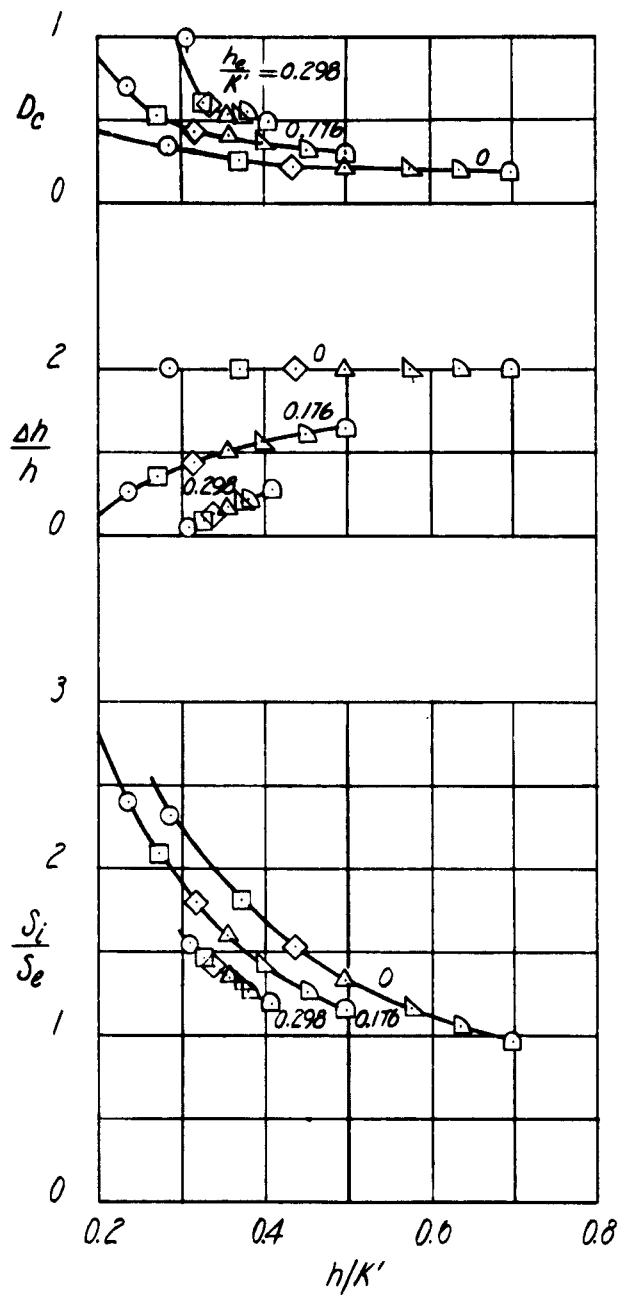


Figure 12 (Concluded)

$$(f) \frac{S}{S_1} = 23.99$$

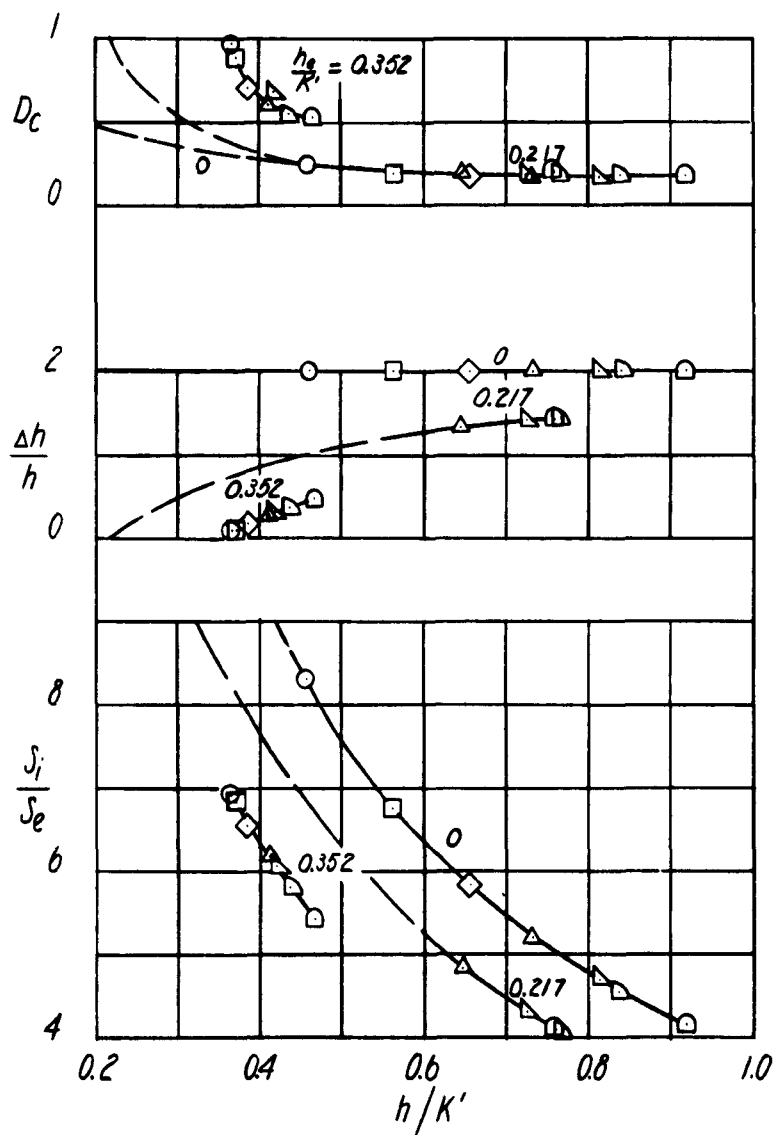


Figure 13 - Variation of the Parameters  $D_c$ ,  $\Delta h/h$ , and  $S_i/S_e$  With  $h/k'$  for Several Values of  $h_0/k'$ .  $\theta = -60^\circ$   
 (a)  $\frac{S}{S_1} = 5.31$

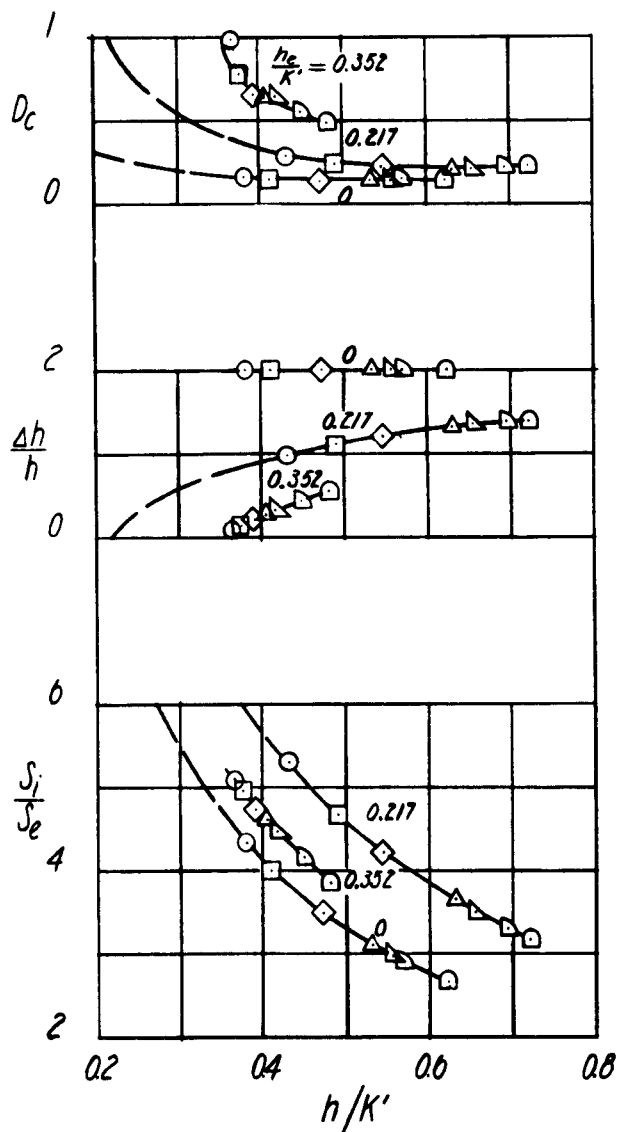


Figure 13 (Continued)

$$(b) \frac{S}{S_1} = 7.23$$

FIGURE 13b

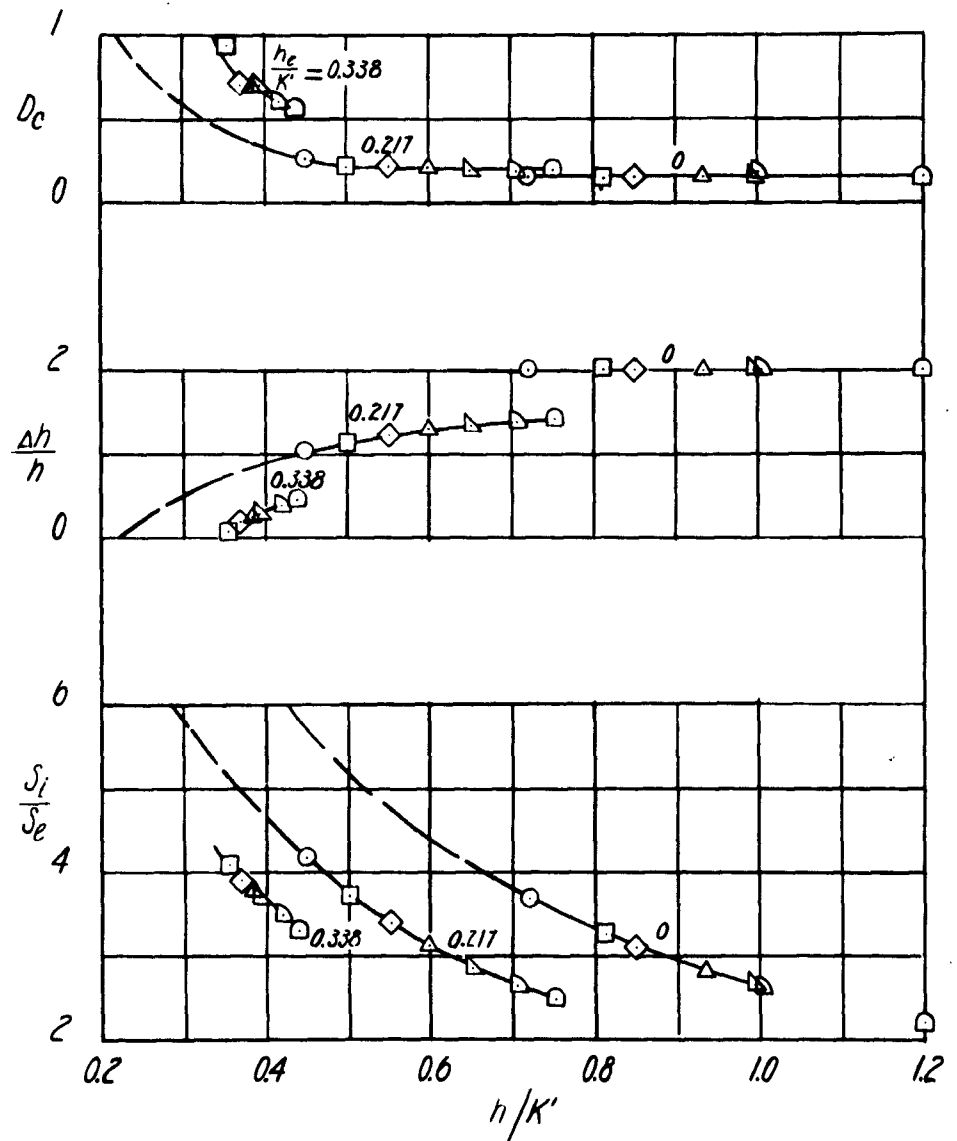


Figure 13 (Continued)

(c)  $\frac{S}{S_1} = 8.88$ 

FIGURE 13c

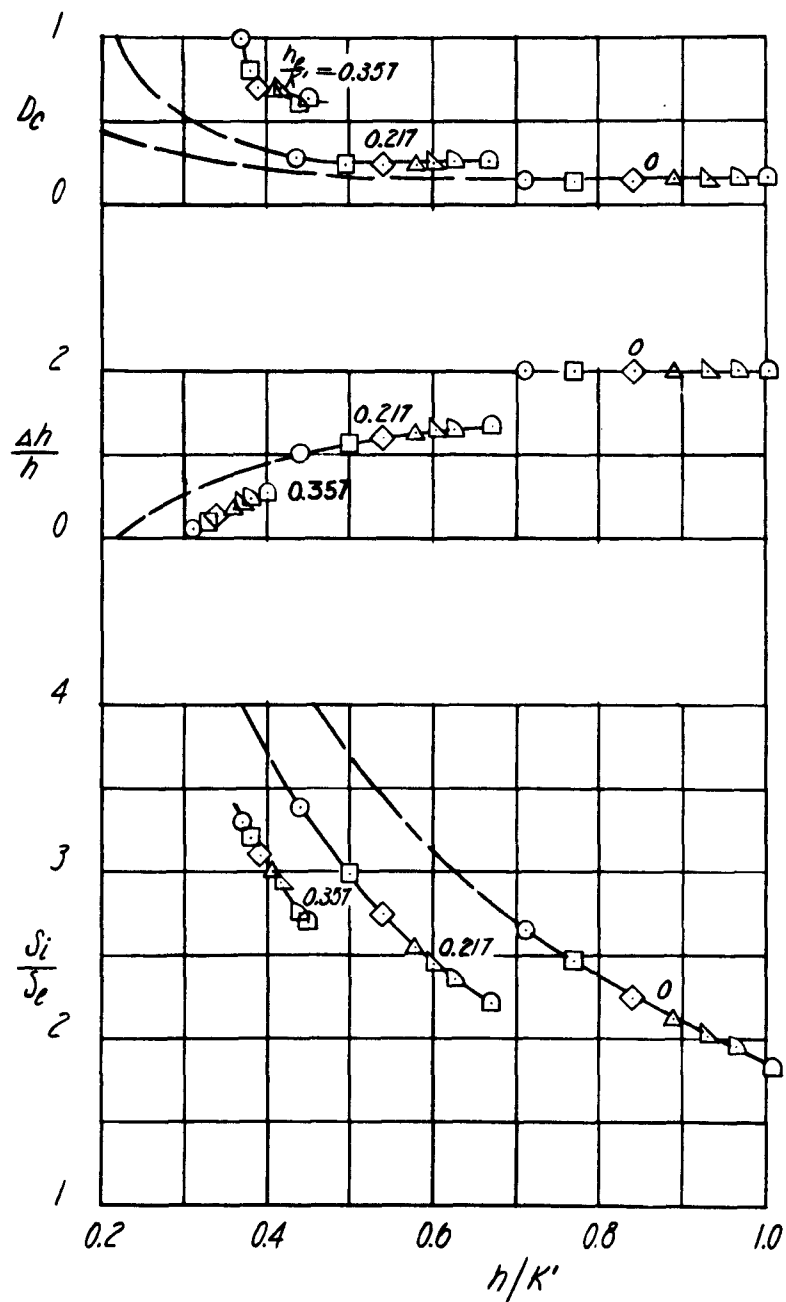


Figure 13 (Continued)

(d)  $\frac{S}{S_1} = 11.20$ 

FIGURE 13d

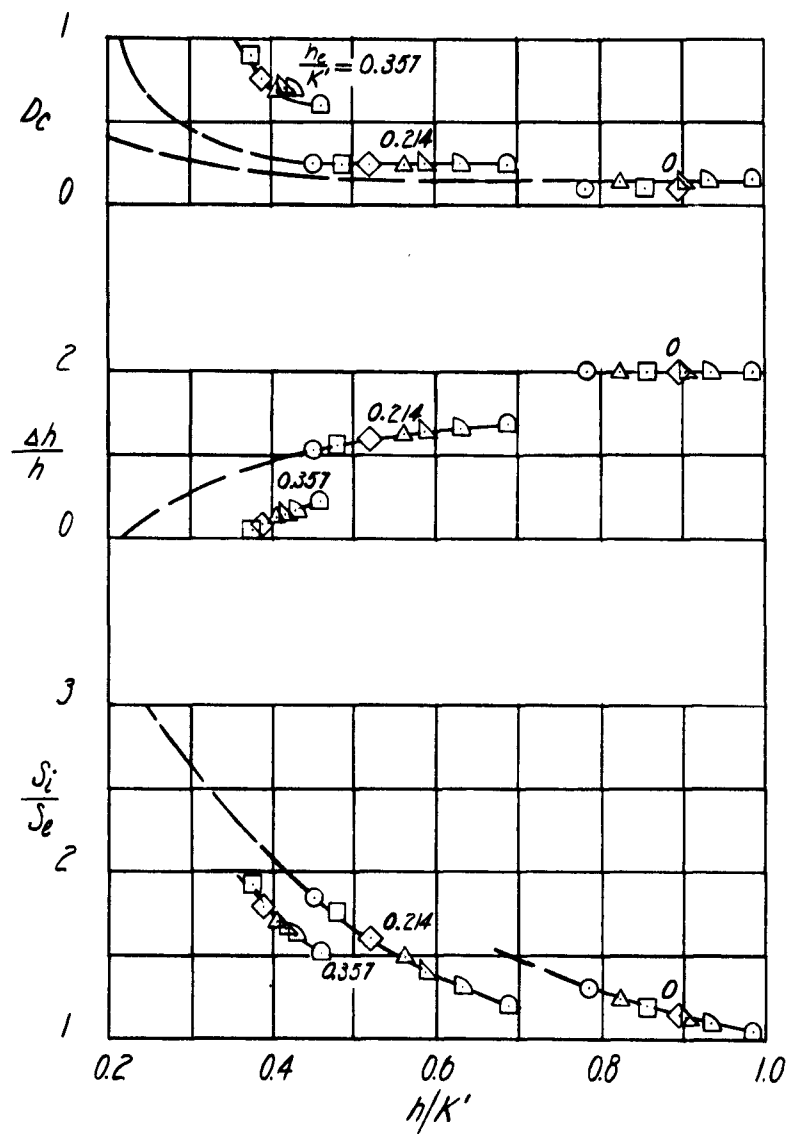


Figure 13 (Continued)

(e)  $\frac{S}{S_1} = 15.27$



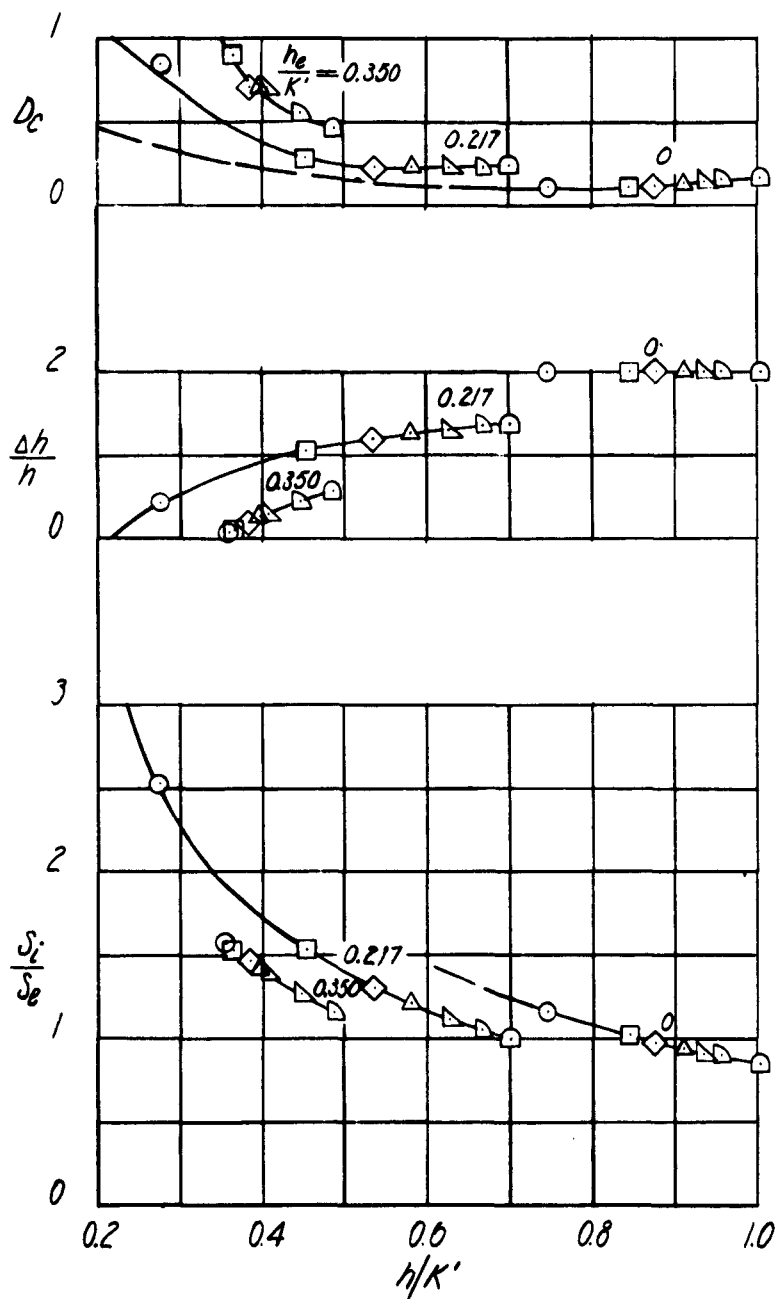


Figure 13 (Concluded)

$$(f) \frac{S}{S_1} = 23.69$$

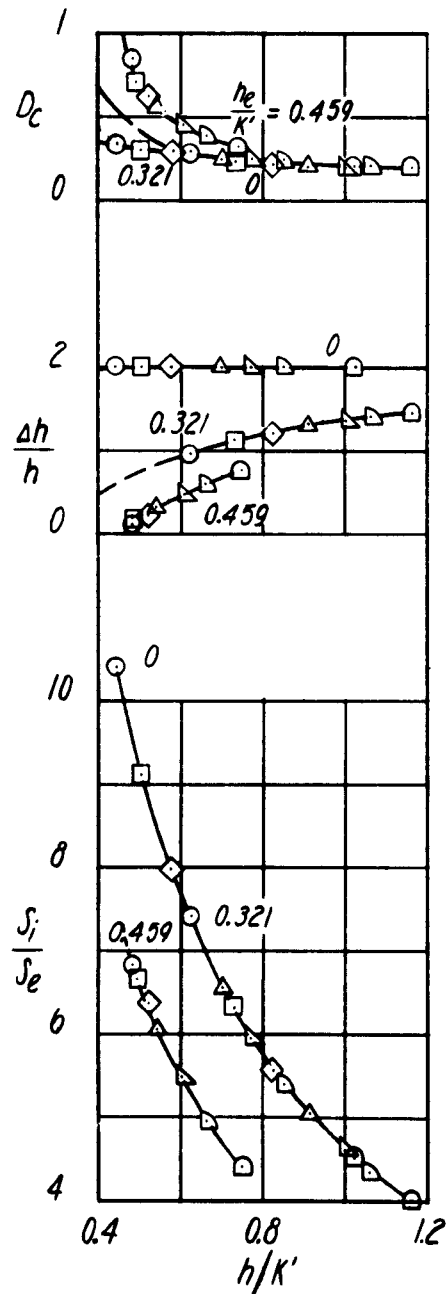


Figure 14 - Variation of the Parameters  $D_c$ ,  $\Delta h/h$ , and  $S_i/S_e$  With  $h/k'$  for Several Values of  $h_e/k'$ .  $\theta = -75^\circ$   
 (a)  $\frac{S}{S_1} = 5.27$

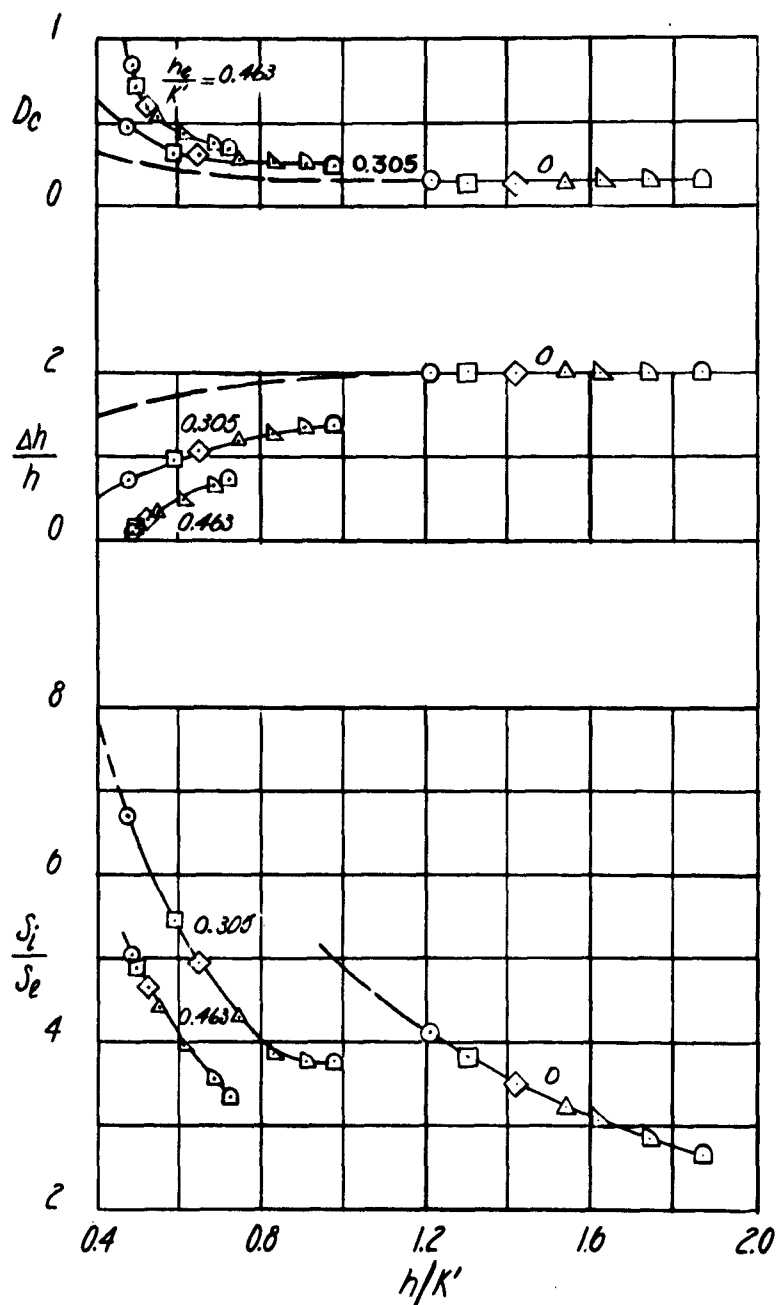


Figure 14 (Continued)

(b)  $\frac{S}{S_1} = 7.17$

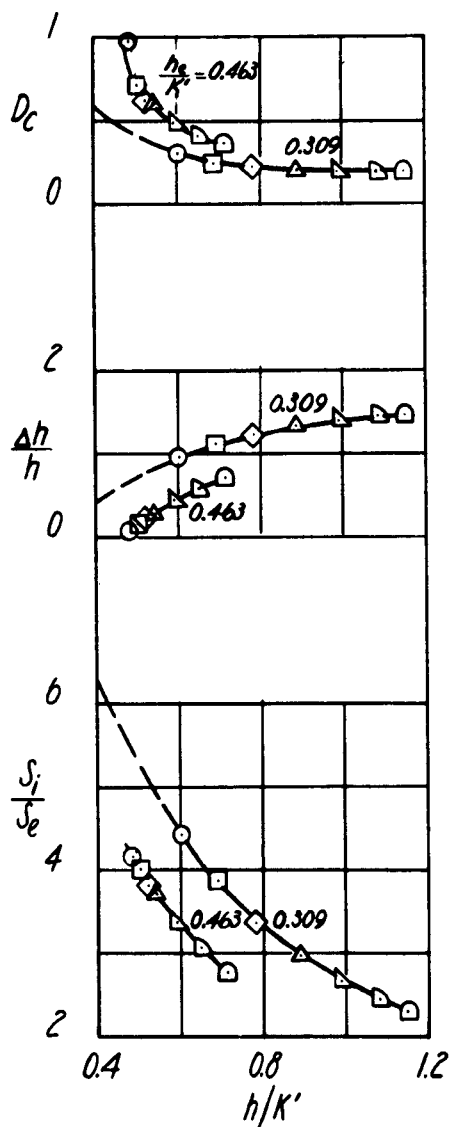


Figure 14 (Continued)

(c)  $\frac{S}{S_1} = 8.81$

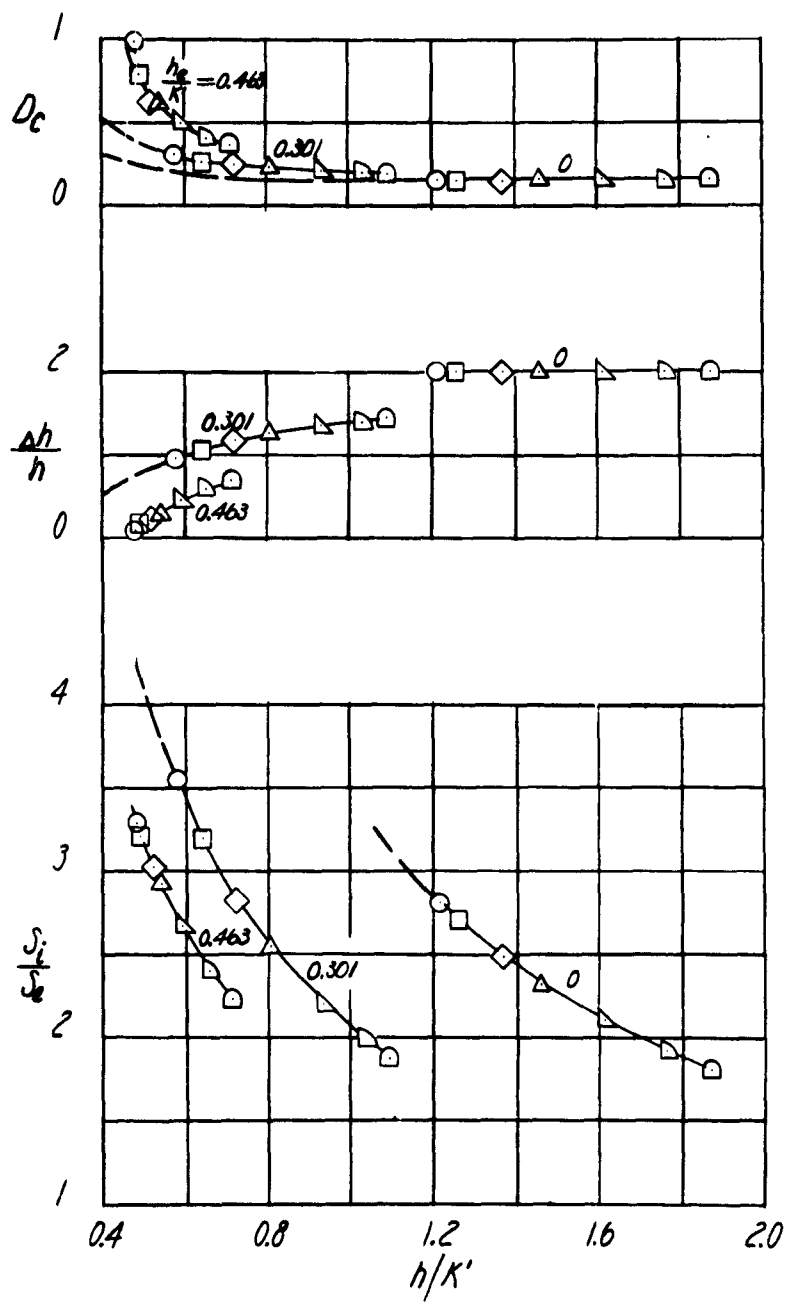


Figure 14 (Continued)

(d)  $\frac{s_2}{s_1} = 11.11$

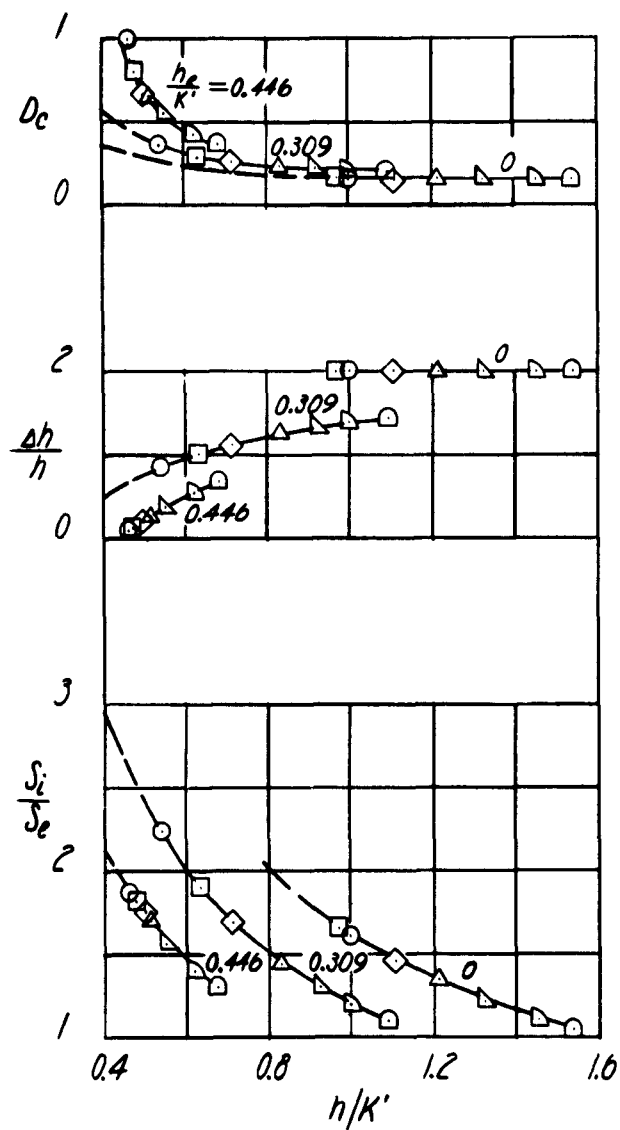


Figure 14 (Continued)

(e)  $\frac{S}{S_1} = 15.15$

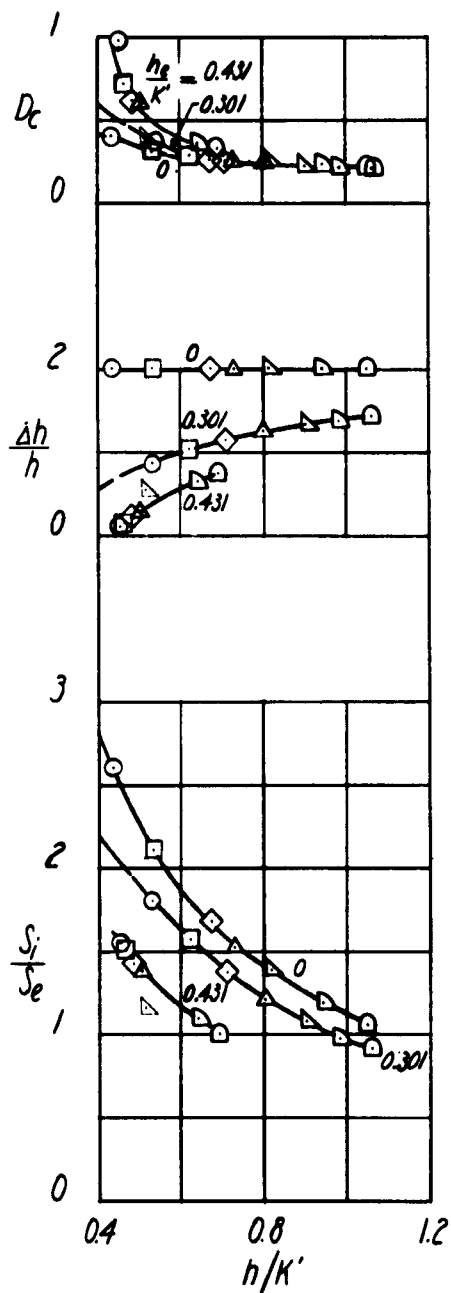


Figure 14 (Concluded)

$$(\epsilon) \frac{S}{S_1} = 23.50$$

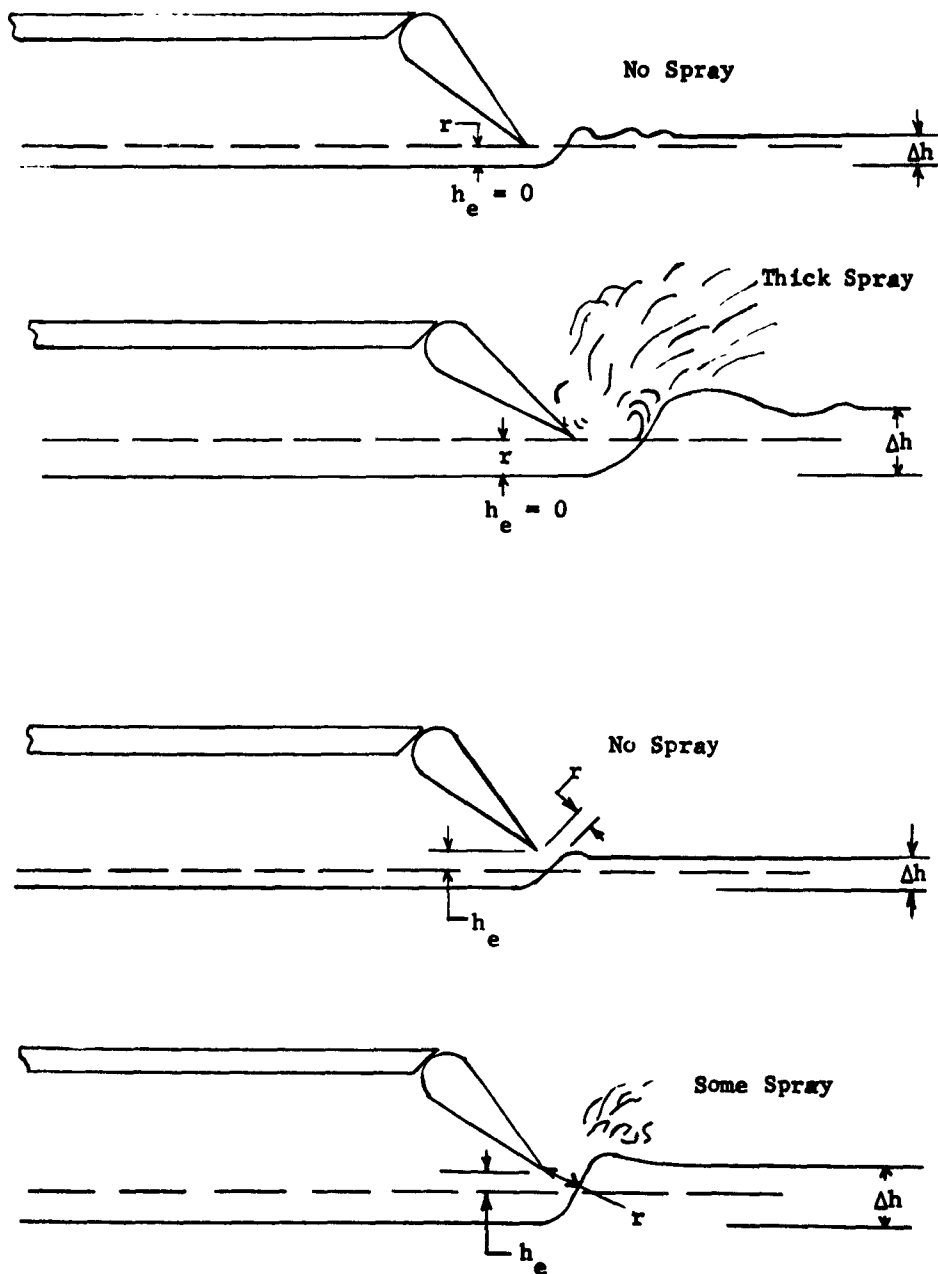


Figure 15 - General Flow Characteristics of the Trailing Edge of the Flap for High and Low Values of  $\Delta h$

(a) Positive  $\theta$



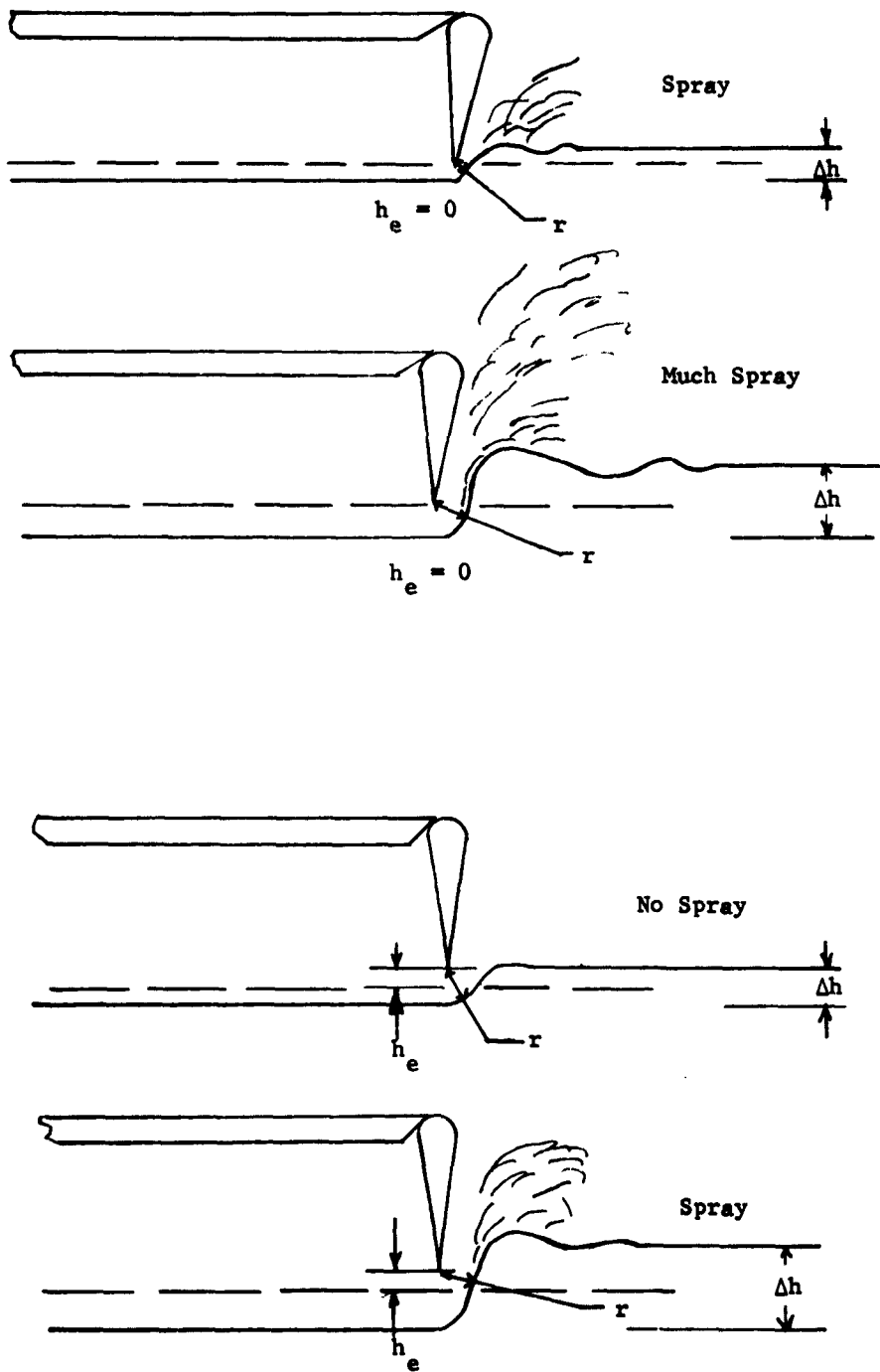


Figure 15 (Continued)  
(b)  $\theta \approx 0^\circ$

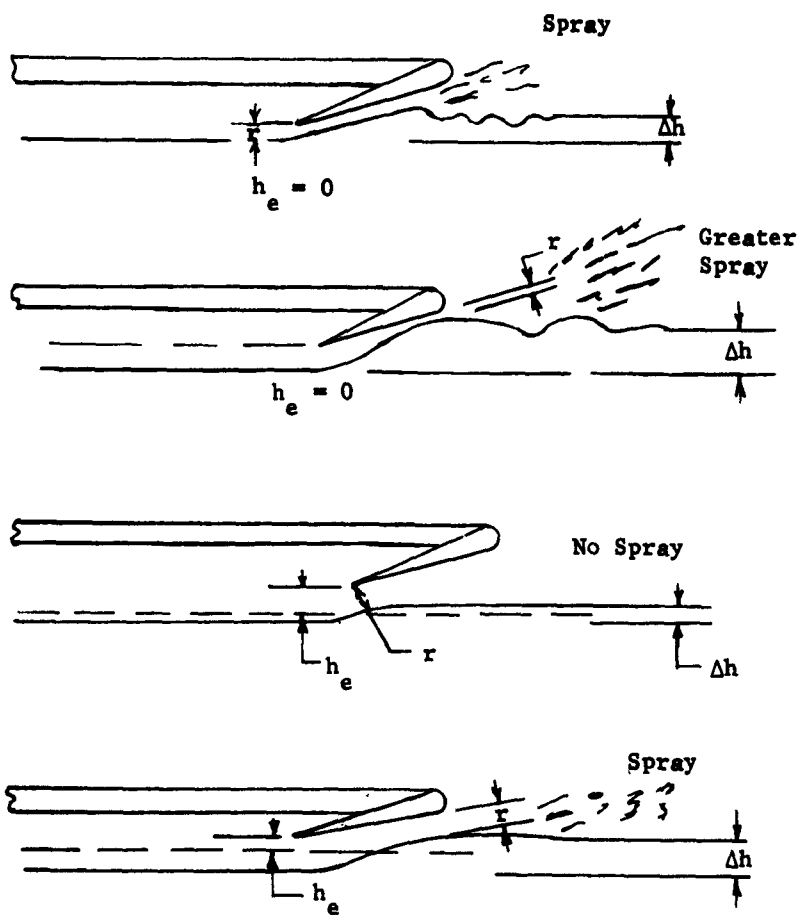


Figure 15 (Concluded)

(c) Negative  $\theta$